

# **SLIDES**SERIES SIP, SHP, SJP & STP RAIL BEARING

#### **Compact Precision**







**Series SJP** 

#### Series STP



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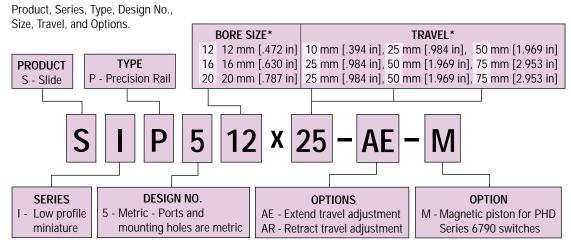
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# ORDERING DATA: SERIES SIP RAIL BEARING SLIDES

#### TO ORDER SPECIFY:



PART NO.	DESCRIPTION
67902-1-02	NPN (Sink) or PNP (Source) DC Reed, 2 m cable
67902-1-05	NPN (Sink) or PNP (Source) DC Reed, 5 m cable
67903-1-02	NPN (Sink) DC Solid State, 2 m cable
67903-1-05	NPN (Sink) DC Solid State, 5 m cable
67904-1-02	PNP (Source) DC Solid State, 2 m cable
67904-1-05	PNP (Source) DC Solid State, 5 m cable
67922-1	NPN (Sink) or PNP (Source) DC Reed, Quick Connect
67923-1	NPN (Sink) DC Solid State, Quick Connect
67924-1	PNP (Source) DC Solid State, Quick Connect
63549-02	2 m Cordset with Quick Connect
63549-05	5 m Cordset with Quick Connect

#### NOTE:

\*Consult PHD for additional bore sizes and travel increments.



CUSTOM SLIDES ARE AVAILABLE. PLEASE CONSULT PHD.



# **BENEFITS:** SERIES SIP RAIL BEARING SLIDES

#### **BENEFITS**

- Series SIP Slides use rail bearing technology to provide smooth, precise movement with high accuracy within confined spaces.
- Available in three bore sizes with a choice of three travel lengths each.
- Standard internal shock pads eliminate metal to metal contact; reducing noise and end-of-travel impact forces.
- Standard Series SIP Slide feature mounting holes on the end and bottom of the body.
- The slide body incorporates switch slots for convenient mounting of PHD's Series 6790 4 mm reed and solid state switches. Magnet option (-M) is required when using Series 6790 Switches.
- Series SIP Slides offer optional 5 mm of travel adjustment on extend (-AE) or retract (-AR). Specify -AE option for extend travel adjustment, -AR for retract travel adjustment, or -AE -AR for both.



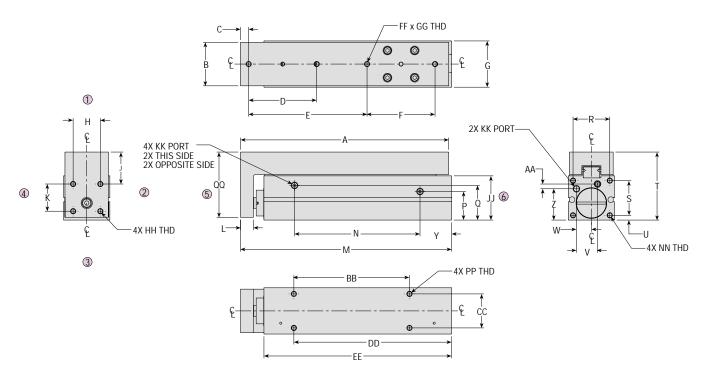
	BORE SIZE					
		12	16		20	
SPECIFICATIONS	in	mm	in	mm	in	mm
POWER SOURCE	Integral pneumatic cylinder					
MAX. OPERATING PRESSURE			100 psi	[7 bar]		
OPERATING TEMPERATURE RANGE	-20° to 180°F [-29° to 82°C]					
TRAVELS	.394	10	.984	25	.984	25
	.984	25	1.969	50	1.969	50
	1.969	50	2.953	75	2.953	75
STANDARD FEATURES		Multiple po	ort positions,	dual mountir	ng positions	
BEARINGS	Stainless s	teel ground i	rail bearing sy	stem with re	ecirculating b	all bearings
BODY	Anodized aluminum alloy					
TOOL PLATE			Anodized alu	ıminum alloy	'	

#### FORCE TABLE

	SIZE 12		SIZE	<u> 16</u>	SIZI	E 20
DIRECTION	lb/psi	N/bar	lb/psi	N/bar	lb/psi	N/bar
EXTEND	.176	11.4	.314	20.3	.486	31.4
RETRACT	.133	8.6	.270	17.4	.409	26.4



# **DIMENSIONS:** SERIES SIP RAIL BEARING SLIDES



		SIPx12			SIPx16			SIPx20	
LETTER		TRAVEL, in [mm]			TRAVEL, in [mm]			TRAVEL, in [mm]	
DIM	.394 [10.0]	.984 [25.0]	1.969 [50.0]	.984 [25.0]	1.969 [50.0]	2.953 [75.0]	.984 [25.0]	1.969 [50.0]	2.953 [75.0]
A	2.953 [75.0]	3.543 [90.0]	4.528 [115.0]	4.016 [102.0]	5.000 [127.0]	5.984 [152.0]	4.055 [103.0]	5.039 [128.0]	6.024 [153.0]
В	2.700 [70.0]	.827 [21.0]	1.020 [1.0.0]	[102.0]	1.063 [27.0]	0.701 [102.0]	1.000 [100.0]	1.260 [32.0]	0.021 [100.0]
С		.197 [5.0]			.236 [6.0]			.236 [6.0]	
D	_		.787 [20.0]	_	1.654	[42.0]		- [	1.969 [50.0]
Е	.827 [21.0]	1.417 [36.0]	2.402 [61.0]	1.614 [41.0]	2.598 [66.0]	3.583 [91.0]	1.457 [37.0]	2.441 [62.0]	3.425 [87.0]
F		1.417 [36.0]			1.654 [42.0]			1.969 [50.0]	
G		.906 [23.0]			1.142 [29.0]			1.339 [34.0]	
Н		.551 [14.0]			.709 [18.0]			.787 [20.0]	
J		.591 [15.0]			.748 [19.0]			.925 [23.5]	
K		.551 [14.0]			.709 [18.0]			.787 [20.0]	
L		.276 [7.0]			.295 [7.5]			.374 [9.5]	
M	3.031 [77.0]	3.622 [92.0]	4.606 [117.0]	4.094 [104.0]	5.079 [129.0]	6.063 [154.0]	4.134 [105.0]	5.118 [130.0]	6.102 [155.0]
N	.776 [19.7]	1.366 [34.7]	2.350 [59.7]	1.764 [44.8]	2.748 [69.8]	3.732 [94.8]	1.661 [42.2]	2.646 [67.2]	3.630 [92.2]
Р		.256 [6.5]			.197 [5.0]			.827 [21.0]	
Q		.673 [17.1]			.866 [22.0]			1.004 [25.5]	
R		.669 [17.0]			.827 [21.0]			1.063 [27.0]	
S		.591 [15.0]			.827 [21.0]			.984 [25.0]	
T		1.299 [33.0]			1.693 [43.0]			1.969 [50.0]	
U		.138 [3.5]			.157 [4.0]			.157 [4.0]	
V W		.276 [7.0]			.413 [10.5]			.610 [15.5]	
Y		.098 [2.5] .839 [21.3]			.256 [6.5] .913 [23.2]			.433 [11.0] .917 [23.3]	
Z		.674 [17.1]			.846 [21.5]			.906 [23.0]	
AA		0.00			.070 [1.8]			.138 [3.5]	
BB	.787 [20.0]	1.378 [35.0]	2.362 [60.0]	1.339 [34.0]	2.323 [59.0]	3.307 [84.0]	1.378 [35.0]	2.362 [60.0]	3.346 [85.0]
CC	.707 [20.0]	.669 [17.0]	2.302 [00.0]	1.557 [54.0]	.866 [22.0]	3.307 [04.0]	1.370 [33.0]	.984 [25.0]	3.340 [03.0]
DD	1.890 [48.0]	2.480 [63.0]	3.465 [88.0]	2.677 [68.0]	3.661 [93.0]	4.646 [118.0]	2.598 [66.0]	3.583 [91.0]	4.567 [116.0]
EE	2.461 [62.5]	3.051 [77.5]	4.035 [102.5]	3.504 [89.0]	4.488 [114.0]	5.472 [139.0]	3.465 [88.0]	4.449 [113.0]	5.433 [138.0]
FF	3	3	4	3	4	4	3	3	4
GG		[M3 x 0.5 x 4.6]			[M4 x 0.7 x 6]		-	[M4 x 0.7 x 6]	
HH		[M3 x 0.5 x 7]			[M4 x 0.7 x 7.5]			[M4 x 0.7 x 9.5]	
JJ		.827 [21.0]			1.102 [28.0]			1.280 [32.5]	
KK		[M3 x 0.5 x 3.5]			[M5 x 0.8 x 4]			[M5 x 0.8 x 4.0]	
NN		[M3 x 0.5 x 5.5]			[M4 x 0.7 x 8]			[M4 x 0.7 x 8]	
PP		[M3 x 0.5 x 5.5]			[M4 x 0.7 x 6]			[M4 x 0.7 x 6]	
QQ		1.260 [32.0]			1.614 [41.0]			1.890 [48.0]	

- NOTES:
  1) DESIGNATED & IS CENTERLINE OF UNIT
  2) METRIC INFORMATION SHOWN IN []
  3) CIRCLED NUMBERS INDICATE POSITION CALLOUT



# ENGINEERING DATA: SERIES SIP RAIL BEARING SLIDES

#### PRESSURE RATINGS

All Series SIP Slides have an operating pressure range of 20 psi minimum to 100 psi maximum [1.4 to 6.9 bar]. Maximum life will be achieved when pressure and velocity are no greater than necessary for proper operation. External flow controls are recommended. Series SIP Slides feature standard pneumatic ports on the end and both sides of the slide body, and are provided with the end ports ready for use and the side ports plugged with set screws and thread sealant.

#### OPERATING TEMPERATURE

Series SIP Slides are designed for use in temperatures between -20° to 180° F [-29° to 82° C]. For temperatures outside this range, consult PHD.

#### **SEALS**

Series SIP Slides utilize urethane and Nitrile seals which are compatible with standard paraffin-based lubrication oils used for pneumatic cylinders. For compatibility with other fluids, consult PHD.

#### LUBRICATION

All units are pre-lubricated at the factory for service under normal operating conditions. Slides are designed and tested with non-lubricated air. However, the use of lubricated air will extend life.

#### MATERIAL SPECIFICATIONS

The slide housing and tool plate are anodized aluminum alloy. Linear rail and bearings are hardened and ground stainless steel.

#### **MAINTENANCE**

Common with most PHD products, these slides are fully field repairable. Repair kits and main structural components are available as needed for extended service life.

#### TOTAL TRAVEL LENGTH AND WEIGHT

For standard units the tolerance of nominal travel lengths is +.039/-.000 [+1 mm/-0 mm]. See Options section of catalog for details regarding units with travel adjustment option (-AE or -AR).

	NOMI TRA		UNIT BASE WEIGHT		
SIZE	in	mm	lb	kg	
	(.394)	10	.30	.14	
12	(.984)	25	.35	.16	
	(1.969)	50	.46	.21	
	(.984)	25	.71	.32	
16	(1.969)	50	.88	.40	
	(2.953)	75	1.04	.47	
	(.984)	25	1.04	.47	
20	(1.969)	50	1.26	.57	
	(2.953)	75	1.48	.67	

#### MOUNTING INSTRUCTIONS

PHD recommends mounting load or tooling with tool plate retracted. Support tool plate while tightening fasteners. Maximum mounting torques (for screw thread engagement of one diameter or greater):

SIZE	in-lb	Nm
M2	2.6	0.3
M3	9	1.0
M4	18	2.0



# **ENGINEERING DATA:** SERIES SIP RAIL BEARING SLIDES

#### **SLIDE SELECTION**

There are three major factors to consider when selecting a slide: thrust capacity, dynamic moment capacity, and the allowable velocity.

#### 1 THRUST CAPACITY

Use the theoretical force output table to determine if thrust is sufficient for the applied load.

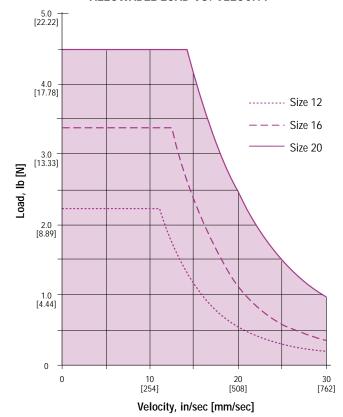
#### 2 DYNAMIC MOMENT CAPACITY

The Dynamic Moment Load graphs (pages 5A-7 to 5A-9) show the allowable load for the three most common mounting positions of the Series SIP Slide. Determine the distance "x" from the edge of the tool plate to the load center of gravity. Use the appropriate graph for the loading condition to determine the allowable load. It is generally best to keep the center of gravity of the load as close to the slide as possible. If the application requires combined loading such as a horizontal pitch load combined with a roll load, if static loads exceed dynamic loads, or if there are other questions concerning the selection of an appropriate slide, please contact PHD's Customer Service Department.

#### 3 ALLOWABLE VELOCITY

Use the allowable velocity graph to verify that the slide selected can carry the payload at the desired velocity.

#### ALLOWABLE LOAD VS. VELOCITY



#### THEORETICAL FORCE OUTPUT TABLE | Ib [N]

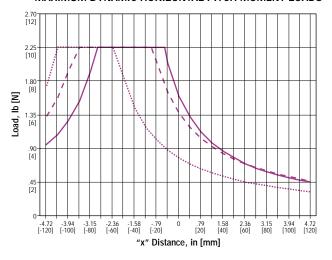
			OPERATING PRESSURE							
		20 psi	30 psi	40 psi	50 psi	60 psi	70 psi	80 psi	90 psi	100 psi
SIZE	DIRECTION	[1.4 bar]	[2.1 bar]	[2.8 bar]	[3.4 bar]	[4.1 bar]	[4.8 bar]	[5.5 bar]	[6.2 bar]	[6.9 bar]
	RETRACT	2.7	4.0	5.3	6.7	8.0	9.3	10.7	12.0	13.3
12	RETRACT	[12.0]	[17.8]	[23.6]	[29.8]	[35.6]	[41.3]	[47.6]	[53.3]	[59.1]
12	FXTFND	3.5	5.3	7.1	8.8	10.6	12.4	14.1	15.9	17.6
	EXTEND	[15.6]	[23.6]	[31.6]	[39.1]	[47.1]	[55.1]	[62.7]	[70.7]	[78.2]
	RETRACT	5.4	8.1	10.8	13.5	16.2	18.9	21.6	24.3	27.0
16		[24.0]	[36.0]	[48.0]	[60.0]	[72.0]	[84.0]	[96.0]	[108.0]	[120.0]
10	FXTFND	6.3	9.4	12.5	15.7	18.8	22.0	25.1	28.2	31.4
	EVIEND	[28.0]	[41.8]	[55.6]	[69.8]	[83.6]	[97.8]	[111.6]	[125.3]	[139.6]
	RETRACT	8.2	12.3	16.4	20.5	24.5	28.6	32.7	36.8	40.9
20	RETRACT	[37.0]	[55.4]	[73.9]	[89.8]	[108.2]	[126.7]	[145.2]	[163.7]	[182.2]
20	EVTEND	9.7	14.6	19.4	24.3	29.2	34.0	38.9	43.7	48.6
	EXTEND	[44.0]	[65.9]	[87.9]	[106.8]	[128.7]	[150.7]	[172.7]	[194.7]	[216.7]

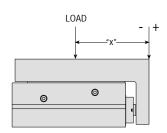


# ENGINEERING DATA: SERIES SIP SIZE 12, DYNAMIC MOMENT LOADS

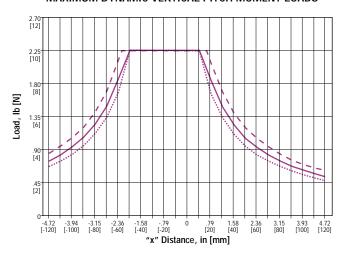
SIP12x10
- - - SIP12x25
SIP12x50

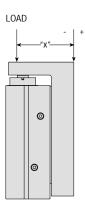
#### MAXIMUM DYNAMIC HORIZONTAL PITCH MOMENT LOADS



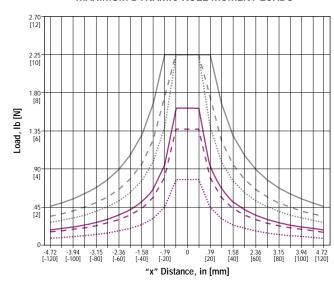


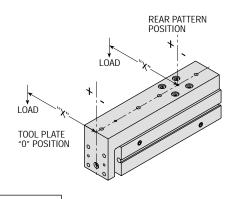
#### MAXIMUM DYNAMIC VERTICAL PITCH MOMENT LOADS





#### MAXIMUM DYNAMIC ROLL MOMENT LOADS





 SIP12x10, tool plate "0"
 SIP12x25, tool plate "0"
 SIP12x50, tool plate "0"
 SIP12x10, rear pattern
 SIP12x25, rear pattern
 SIP12x50, rear pattern

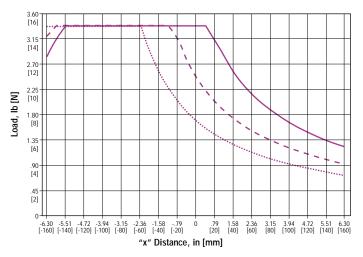
SIP12x10

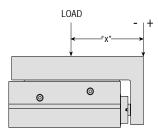
SIP12x25 SIP12x50



# ENGINEERING DATA: SERIES SIP SIZE 16, DYNAMIC MOMENT LOADS

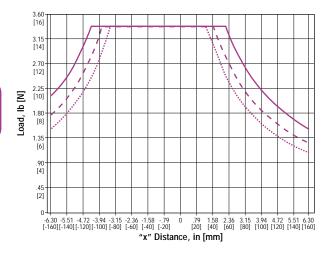
#### MAXIMUM DYNAMIC HORIZONTAL PITCH MOMENT LOADS





SIP16x25
--- SIP16x50
--- SIP16x75

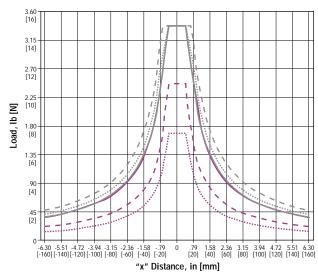
#### MAXIMUM DYNAMIC VERTICAL PITCH MOMENT LOADS

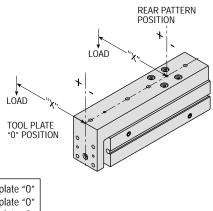


SIP16x25
--- SIP16x50
---- SIP16x75

# 

#### MAXIMUM DYNAMIC ROLL MOMENT LOADS



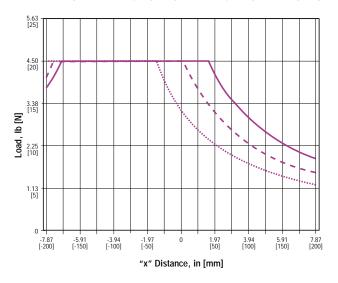


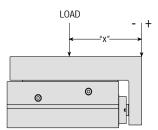
SIP16x25, tool plate "0"
- - - SIP16x50, tool plate "0"
SIP16x75, tool plate "0"
SIP16x25, rear pattern
- - - SIP16x50, rear pattern
SIP16x75, rear pattern



# ENGINEERING DATA: SERIES SIP SIZE 20, DYNAMIC MOMENT LOADS

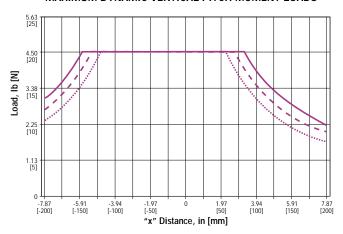
#### MAXIMUM DYNAMIC HORIZONTAL PITCH MOMENT LOADS





SIP20x25
--- SIP20x50
SIP20x75

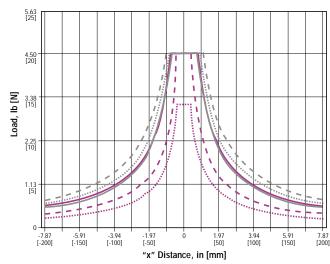
#### MAXIMUM DYNAMIC VERTICAL PITCH MOMENT LOADS

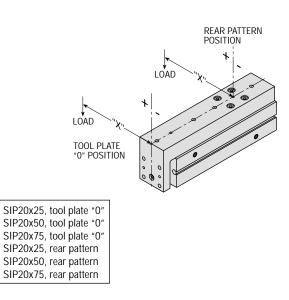




# LOAD "X" — T

# MAXIMUM DYNAMIC ROLL MOMENT LOADS





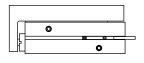


#### MAGNET FOR PHD SERIES 6790 REED AND SOLID STATE SWITCHES

This option equips the unit with a magnetic piston for use with PHD's Series 6790 Switch. The switch housing is contained by the slide housing and provides a very compact switch design. The switches mount easily into two small grooves located on the side of the slide housing and are locked into place with a set screw.

LETTER	SIZE 12		SIZE	16	SIZE 20		
DIM	in	mm	in	mm	in	mm	
Α	0.689	17.5	0.933	23.7	1.122	28.5	
В	0.492	12.5	0.551	14	0.591	15	







#### **SERIES 6790 REED SWITCHES**

	REPEATABILITY		HYSTERESIS MAXIMUM		BANDWIDTH MIN./MAX.		
SIZE	in	mm	in	mm	in	mm	
12	+/005	+/13	0.060	1.5	.380/.400	9.7/10.2	
16	+/005	+/13	0.080	2.0	.220/.350	5.6/8.9	
20	+/005	+/13	0.060	1.5	.335/.750	8.5/19.0	

#### **SERIES 6790 SOLID STATE SWITCHES**

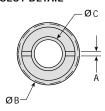
	REPEATABILITY		REPEATABILITY HYSTERESIS MAXIMUM		BANDWIDTH MIN./MAX.		
SIZE	in	mm	in	mm	in	mm	
12	+/005	+/13	0.060	1.5	.335/.630	8.5/16.0	
16	+/005	+/13	0.080	2.0	.170/.490	4.3/12.4	
20	+/005	+/13	0.060	1.5	.280/.670	7.1/17.0	



# TRAVEL ADJUSTMENT ON EXTEND

This option provides up to 5 mm of travel on extend. Travel adjustment is made using a spanner wrench or similar tool to engage the slots in the cartridge and rotating the cartridge to the desired position. Rotating the cartridge clockwise reduces the travel. Normal shock pad operation is maintained regardless of cartridge position. Travel adjustment has internal stops, preventing loss of components. The -AE option may be used in conjunction with the -AR option to provide travel adjustment at both ends of travel.

#### **CARTRIDGE SLOT DETAIL**



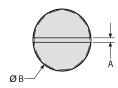
	"A" SLOT WIDTH		MAX TO	3" OOL DIA		ROD NCE DIA	SLOT	DEPTH
SIZE	in	mm	in	mm	in	mm	in	mm
12	.062	1.6	.450	11.4	.215	5.5	.030	.8
16	.062	1.6	.600	15.2	.362	9.2	.060	1.5
20	.062	1.6	.817	20.8	.478	12.1	.060	1.5



# TRAVEL ADJUSTMENT ON RETRACT

This option provides up to 5 mm of travel on retract. Travel adjustment is made using a flat-bladed screwdriver or similar tool to engage the slot in the bore plug and rotating the bore plug to the desired position. Rotating the bore plug clockwise reduces the travel. Normal shock pad operation is maintained regardless of bore plug position. Travel adjustment has internal stops, preventing loss of components. The -AR option may be used in conjunction with the -AE option to provide travel adjustment at both ends of travel.

#### **BORE PLUG SLOT DETAIL**



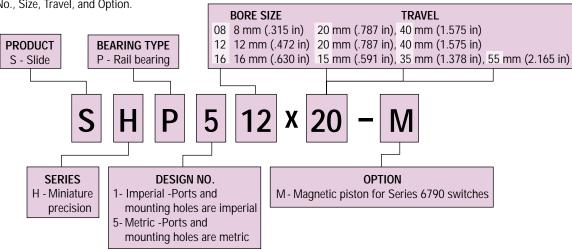
	"A" SLO	OT WIDTH	"B" MAX	TOOL DIA	SLOT DEPTH	
SIZE	in	mm	in	mm	in	mm
12	.062	1.6	.450	11.4	.030	.8
16	.062	1.6	.600	15.2	.060	1.5
20	062	1.6	817	20.8	060	1.5





#### TO ORDER SPECIFY:

Product, Series, Bearing Type, Design No., Size, Travel, and Option.



ORDERING DATA: SERIES SHP SLIDES WITH RAIL BEARING

PART NO.	DESCRIPTION
67902-1-02	NPN (Sink) or PNP (Source) DC Reed, 2 m cable
67902-1-05	NPN (Sink) or PNP (Source) DC Reed, 5 m cable
67903-1-02	NPN (Sink) DC Solid State, 2 m cable
67903-1-05	NPN (Sink) DC Solid State, 5 m cable
67904-1-02	PNP (Source) DC Solid State, 2 m cable
67904-1-05	PNP (Source) DC Solid State, 5 m cable
67922-1	NPN (Sink) or PNP (Source) DC Reed, Quick Connect
67923-1	NPN (Sink) DC Solid State, Quick Connect
67924-1	PNP (Source) DC Solid State, Quick Connect
63549-02	2 m Cordset with Quick Connect
63549-05	5 m Cordset with Quick Connect

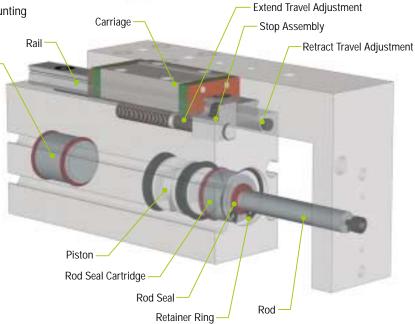
# **BENEFITS:** SERIES SHP SLIDES WITH RAIL BEARING

#### **BENEFITS**

- Series SHP Slides use rail bearing technology to provide smooth, precise movement within confined spaces.
- Available in three bore sizes and five travel lengths.
- Standard travel adjustments for both extend and retract positions are conveniently located on the back of the slide. Integrated shock pads provide shock reduction throughout the full range of travel adjustment.
- Available in both imperial and metric versions for applications in worldwide markets.
- Standard Series SHP Slides provide multiple mounting options for maximum flexibility. Bodies feature mounting holes on sides, end, and bottom. Tool plates feature mounting on top and end.

  Bore Plug
- Standard dowel holes are provided on the body and both end and top tool plate mounting positions for precise mounting and attachment of tooling.
- Slide housing provides dual switch slots on either side for convenient mounting of PHD's Series 6790 4 mm Reed and Solid State switches. Magnet option (-M) is required when using switches.





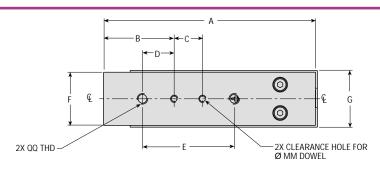
			BORE	SIZE				
		08	1	2	16			
SPECIFICATIONS	in	mm	in	mm	in	mm		
POWER SOURCE	Integral pneumatic cylinder							
MAX. OPERATING PRESSURE	100 psi [6.9 bar]							
OPERATING TEMPERATURE RANGE	-20° to 180°F [-29° to 82°C]							
TRAVELS	.79	20	.79	20	.59	15		
	1.57	40	1.57	40	1.38	35		
					2.17	55		
TRAVEL ADJUSTMENTS		Stan	dard on both	extend and r	etract			
STANDARD FEATURES	Multiple	port position	ıs, multiple n	nounting posi	itions, dowe	pin holes		
BEARINGS	Stainless s	steel ground r	ail bearing s	ystem with re	ecirculating b	all bearings		
BODY	Anodized aluminum alloy							
TOOL PLATE			Anodized alu	uminum alloy	1			

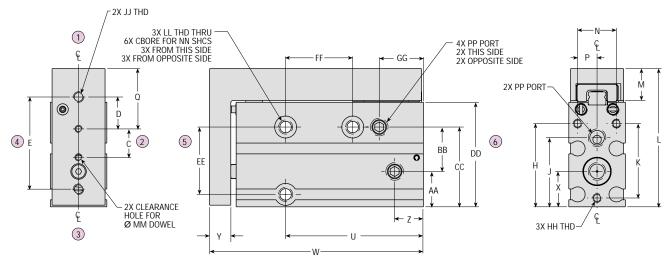
#### FORCE TABLE

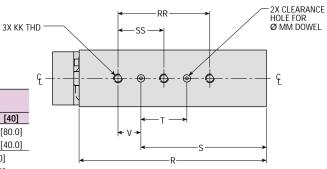
DIRECTION	SIZE 08 Ib/psi N/B		SIZE Ib/psi	12 N/B	SIZE 16 lb/psi N/B		
EXTEND	.079	5.1	.176	11.4	.314	20.3	
RETRACT	.060	3.9	.133	8.6	.270	17.4	



# **DIMENSIONS:** SERIES SHP SLIDES - SIZE 08







LETTER		SIZE 08									
DIM.	0.79	TRAVEL, [20]	in [mn 1.57								
				[40]							
Α	2.362	[60.0]	3.149	[80.0]							
В	0.787	[20.0]	1.574	[40.0]							
C*	0.	315	[8	3.0]							
D	0.	354	[9.0]								
Ε	1.	024	[26.0]								
F	0.	591	[1	5.0]							
G	0.	630	[1	6.0]							
Н	0.	925	[2	3.5]							
J	0.	767	[19.5]								
K	0.	827	[2	1.0]							
L	1.	534	[3	9.0]							
М	0.	354	[9	0.0]							
N	0.	433	[11.0]								
Р	0.	217	[5.5]								
Q	0.	669	[1]	7.0]							
R	2.087	[53.0]	2.874	[73.0]							
S	1.398	[35.5]	1.791	[45.5]							
T*	0.	512	[1:	3.0]							
U	1.536	[39.0]	2.323	[59.0]							
V	0.	256	[6	.5]							
W	2.383	[60.5]	3.17	[80.5]							
Χ	0.	393	[1	0.0]							
Υ	0.	236	[6.0]								
Z	0.	320	[8	3.1]							

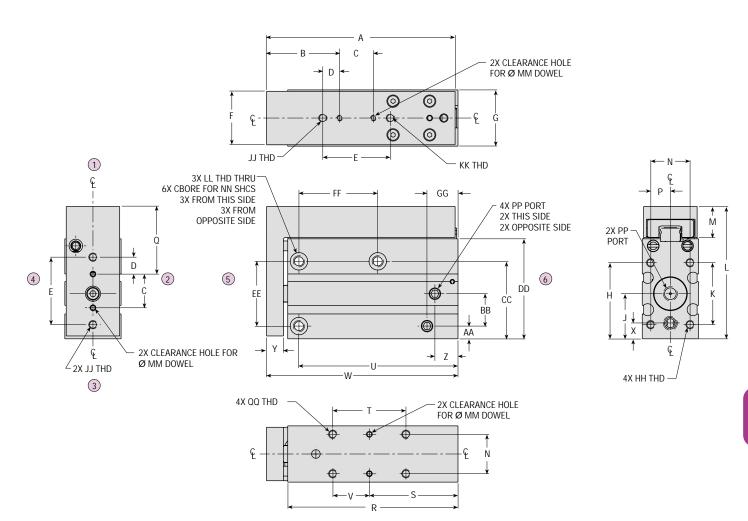
<sup>\*</sup> TOLERANCE IS ± .001 BETWEEN DOWEL PIN HOLES.

	SIZ	'E 08				
LETTER	TRAVEL,	in [mm]				
DIM.	0.79 [20]	1.57 [40]				
AA	.393	[10.0]				
BB	.493	[12.5]				
CC	.886	[22.5]				
DD	1.161	[29.5]				
EE	.748	[19.0]				
FF	0.748 [19.0]	1.535 [39.0]				
GG	0.492	[12.5]				
HH	#4-40 UNC x .320 DP	[M2.5 x 0.45 x 8.1 mm DP]				
JJ	#4-40 UNC THRU	[M3 x 0.5 THRU]				
KK	#4-40 UNC x .140 DP	[M2.5 x 0.45 x 3.5 mm DP]				
LL	#8-32 UNC THRU	[M4 x 0.7 THRU]				
MM	1/16 x .125 DP	[2 mm x 3.1 mm DP]				
NN	#5	[M3]				
PP	#10-32 PORT	[M5 x 0.8 PORT]				
QQ	#4-40 UNC x .157 DP	[M3 x 0.5 x 4.0 mm DP]				
RR	1.024	[26.0]				
SS	.512	[13.0]				

- 1) DESIGNATED & IS CENTERLINE OF UNIT.
- 2) METRIC INFORMATION SHOWN IN [].
- 3) CIRCLED NUMBERS INDICATE POSITION CALLOUT.



# **DIMENSIONS:** SERIES SHP SLIDES - SIZES 12 & 16



		SIZ	E 12				SIZE	16			
LETTER	T	RAVEL,	in [m	m]		TF	RAVEL,	in [mn	n]		
DIM.	0.79	[20]	1.57	[40]	0.59	[15]	1.38	[35]	2.17	[55]	
Α	2.638	[67.0]	3.425	[87.0]	2.874	[73.0]	3.661	[93.0]	4.449	[113.0]	
В	1.024	[26.0]	1.811	[46.0]	1.201	[30.5]	1.988	[50.5]	2.776	[70.5]	
C*	0.4	472	[12.0]			0.4	172	[12	2.0]		
D	0.2	236	[6	.0]		0.5	512	[13	3.0]		
E	0.945		[24	4.0]		1.4	196	[38	3.0]		
F	0.748		[19	9.0]		0.9	945	[24	.0]		
G	0.787		[20	0.0]		0.9	984	[25	5.0]		
Н	1.064		[2]	7.0]	1.141			[29	0.0]		
J	0.631		[16	5.0]	0.669			[17.0]			
K	0.866		[22	2.0]	0.944			[24	[24.0]		
L	1.851		[4]	7.0]	2.106 [53.5			3.5]			
M	0.4	433	[11	1.0]		0.5	551	[14	.0]		
N*	0.!	551	[14	4.0]	0.630			[16.0]			
P	0.2	276	[7.0]		0.315			[8.0]			
Q	_	945		4.0]	.768		[19.5]				
R	2.382	[60.5]	3.169	[80.5]	2.559	[65.0]	3.346	[85.0]	4.134	[105.0]	
S	1.240	[31.5]	1.633	[41.5]	1.338	[34.0]	1.731	[44.0]	2.126	[54.0]	
T		024		5.0]			81	[30			
U	2.225	[56.5]	3.012	[76.5]	2.362	[60.0]	3.149	[80.0]	3.937	[100.0]	
V	0.512 [13.0]				91	[15.0]					
W	2.678	[68.0]	3.465	[88.0]	2.895	[73.5]	3.682	[93.5]	4.47	[113.5]	
Х	l	218	[5	.5]	0.197		[5.0]				
Υ		236	[6	.0]		0.2	276	[7.	.0]		
Z	0.:	323	[8	.2]		0.3	372	[9	.4]		

		SIZ	ĽE 12		SIZE 16						
LETTER		TRAVEL	in [mm]			TI	RAVEL,	in [mr	n]		
DIM.	0.79	[20]	1.57	[40]	0.59	[15]	1.38	[35]	2.17	[55]	
AA	0.1	78	[4.5	5]		0.1	97	[5.	0]		
BB	0.453		[11.	5]		0.4	72	[12	.0]		
CC	1.084		[27.	5]		1.1	61	[29	.5]		
DD	1.399		[35.	5]		1.5	35	[39	.0]		
EE	0.906		[23.0]			0.984		[25.0]			
FF	1.102	[28.0]	1.889	[48.0]	1.181	[30.0]	1.968	[50.0]	2.756	[70.0]	
GG	0.4	133	[11.	0]		0.5	91	[15	.0]		
HH	#4-40 UNC	x .236 DP	[M3 x 0.5 x 6 mm DP]		#8-32 UNC x .276 DP		[M4 x	0.7 x 7 r	nm DP]		
JJ	#4-40 UNC	x .236 DP	[M3 x 0.5 x	6 mm DP]	#8-32 UNC x .295 DP		[M4 x 07 x 7.5 mm DP		mm DP]		
KK	#4-40 UNC	x .167 DP	[M3 x 0.5 x 4	.2 mm DP]	#8-32 UNC x .207 DP		[M4 x 0.7 x 5.2 mm DP]		mm DP]		
LL	#8-32 UI	NC THRU	[M4 X 0.7	THRU]	#10-	-32 UNF 1	HRU	[M5	x 0.8 TI	HRU]	
MM	1/16 x .	125 DP	[2 mm x 3.1	I mm DP]	1/	8 x .188	DP	[2 m	m x 3 m	m DP]	
NN	#	5	[M3]			#6		[M4]			
PP	#10-32	2 PORT	[M5 x 0.8	PORT]	#	#10-32 PORT		[M5 x 0.8 PORT]			
QQ	#4-40 UNC	x .177 DP	[M3 x 0.5 x 4	.5 mm DP]	#8-32	UNC x .2	36 DP	[M4 x	[M4 x 0.7 x 6 mm DF		

#### NOTES

- 1) DESIGNATED & IS CENTERLINE OF UNIT.
- 2) METRIC INFORMATION SHOWN IN [].
- 3) CIRCLED NUMBERS INDICATE POSITION CALLOUT.

<sup>\*</sup> TOLERANCE IS ± .001 BETWEEN DOWEL PIN HOLES.



# **ENGINEERING DATA:** SERIES SHP SLIDES

#### PRESSURE RATINGS

All Series SHP Slides have an operating pressure range of 20 psi minimum to 100 psi maximum [1.4 to 6.9 bar]. For longest slide life it is recommended that pressure and velocity be no greater than necessary for proper operation. Series SHP Slides incorporate internal orifices to help limit velocities. However, external flow controls are recommended at higher pressures or if slide operation results in strong impact loads at ends of travel. Series SHP Slides feature standard pneumatic ports on the end and both sides of the slide body, and are provided with the end ports ready for use and the side ports plugged with set screws and thread sealant.

#### OPERATING TEMPERATURE

Series SHP Slides are designed for use in temperatures between 20° to 180° F [-6° to 82° C]. For temperatures outside this range, consult PHD.

#### **SEALS**

Series SHP Slides utilize urethane and Nitrile seals which are compatible with standard paraffin-based lubrication oils used for pneumatic cylinders. For compatibility with other fluids, consult PHD.

#### LUBRICATION

All units are pre-lubricated at the factory for service under normal operating conditions. Slides are designed and tested with non-lubricated air. However, the use of lubricated air will extend life.

#### MATERIAL SPECIFICATIONS

The slide housing and tool plate are anodized aluminum alloy. Linear rail and bearings are hardened and ground stainless steel.

#### **MAINTENANCE**

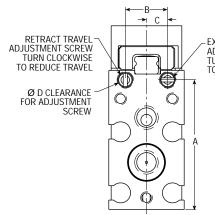
In common with most PHD products, these slides are fully field repairable. Repair kits and main structural components are available as needed for extended service life.

UNIT WEIGHT									
	TRA	VEL	WE	IGHT					
SIZE	in	mm	lb	kg					
	.79	20	.20	.09					
80	1.57	40	.26	.12					
	.79	20	.38	.17					
12	1.57	40	.48	.22					
	.59	15	.56	.25					
16	1.38	35	.71	.32					
	2.17	55	.85	.39					

#### TRAVEL ADJUSTMENT

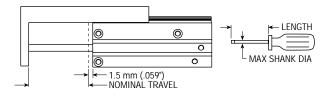
Standard Series SHP Slides provide travel adjustment in both the retract and extend directions. Travel adjustments are made using a small flat bladed or standard screwdriver via the adjustment holes located on the back of the slide. Series SHP Slides are designed to provide nominal travel. Using the travel adjustment screws allows reducing either the extend or retract travel by .394 in [10 mm] (5 mm for SHP08).

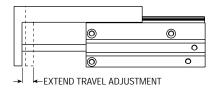
Travel adjustment requires a small flat bladed screwdriver with a minimum shank length and diameter as shown in the table below. Blade thickness should not exceed .030 in [.75 mm]. Travel adjustments should not be adjusted beyond positions shown in illustration. Loss of components or damage to the mechanism may occur if adjusted beyond the recommended limits.

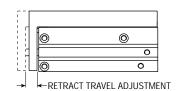


- EXTEND TRAVEL ADJUSTMENT SCREW TURN COUNTER-CLOCKWISE TO REDUCE TRAVEL

#### TRAVEL ADJUSTMENT







				- 1						
	SIZE									
LETTER	0	8	1:	2	1	6				
DIM	in	mm	in	mm	in	mm				
Α	1.082	27.5	1.300	33.0	1.436	36.5				
В	0.354	9.0	0.480	12.2	0.570	14.5				
С	0.177	4.5	0.240	6.1	0.285	7.2				
D	0.125	3.2	0.165	4.2	0.165	4.2				

		NOMINAL	TRAVEL	EXTEND	TRAVEL	RETRACT TRAVEL		ADJUSTMENT MIN.		SCREWDRIVER MAX.		
				ADJUS	ADJUSTMENT		ADJUSTMENT		SHANK LENGTH		SHANK DIAMETER	
	SIZE	in	mm	in	mm	in	mm	in	mm	in	mm	
		0.79	20	.197	5	.197	5	1.5	38	.083	2.1	
	08	1.57	40	.197	5	.197	5	2.3	58	.083	2.1	
	40	0.79	20	.394	10	.394	10	1.1	28	.130	3.3	
	12	1.57	40	.394	10	.394	10	1.2	30	.130	3.3	
ı		0.59	15	.394	10	.394	10	1.3	33	.130	3.3	
	16	1.38	35	.394	10	.394	10	1.3	33	.130	3.3	
		2.17	55	.394	10	.394	10	2.2	55	.130	3.3	

# ENGINEERING DATA: SERIES SHP SLIDES

#### **SLIDE SELECTION**

There are three major factors to consider when selecting a slide: thrust capacity, allowable mass, and dynamic moment capacity.

#### 1 THRUST CAPACITY

Use the theoretical output table to determine if thrust is sufficient for the applied load.

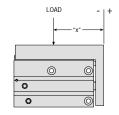
#### MAXIMUM PAYLOAD CAPACITY

All Series SHP Slides come standard with end of travel shock pads. However, these shock pads are limited in the amount of energy that they can dissipate. Therefore, the slides have a maximum payload limit. Use the allowable velocity graph to verify that the slide can carry the payload at the desired velocity.

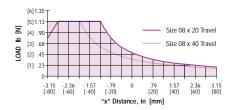
#### DYNAMIC MOMENT CAPACITY

The Dynamic Moment Load graphs show the allowable load for the three most common mounting positions of the Series SHP Slide. Determine the distance "x"

from the edge of the tool plate to the load center of gravity. Use the graph appropriate for the loading condition to determine the allowable load. It is generally best to keep the load center of gravity as close to the slide as possible. (See next page for graphs.) If the application requires combined loading such as a horizontal pitch load combined with a roll load, if static loads exceed dynamic loads, or if there are other questions concerning the selection of an appropriate slide, please contact PHD's Customer Service Department.



SIZE 08



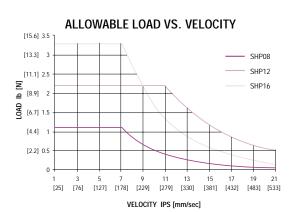
#### THEORETICAL OUTPUT TABLE Ib [N]

					OPER#	TING PRES	SSURE			
		20 psi	30 psi	40 psi	50 psi	60 psi	70 psi	80 psi	90 psi	100 psi
SIZE	DIRECTION	[1.4 bar]	[2.1 bar]	[2.8 bar]	[3.4 bar]	[4.1 bar]	[4.8 bar]	[5.5 bar]	[6.2 bar]	[6.9 bar]
	RETRACT	1.2	1.8	2.4	3.0	3.6	4.2	4.8	5.4	6.0
08	KLIKACI	[5.3]	[8.0]	[10.7]	[13.3]	[16.0]	[18.7]	[21.3]	[24.0]	[26.7]
08	FXTFND	1.6	2.4	3.1	3.9	4.7	5.5	6.3	7.1	7.9
	EVIEND	[7.1]	[10.7]	[13.8]	[17.3]	[20.9]	[24.4]	[28.0]	[31.6]	[35.1]
	RETRACT	2.7	4.0	5.3	6.7	8.0	9.3	10.7	12.0	13.3
12		[12.0]	[17.8]	[23.6]	[29.8]	[35.6]	[41.3]	[47.6]	[53.3]	[59.1]
12	FXTFND	3.5	5.3	7.1	8.8	10.6	12.4	14.1	15.9	17.6
	LATEND	[15.6]	[23.6]	[31.6]	[39.1]	[47.1]	[55.1]	[62.7]	[70.7]	[78.2]
	RETRACT	5.4	8.1	10.8	13.5	16.2	18.9	21.6	24.3	27.0
16	KEIKACI	[24.0]	[36.0]	[48.0]	[60.0]	[72.0]	[84.0]	[96.0]	[108.0]	[120.0]
	EXTEND	6.3	9.4	12.5	15.7	18.8	22.0	25.1	28.2	31.4
	EXTEND	[28.0]	[41.8]	[55.6]	[69.8]	[83.6]	[97.8]	[111.6]	[125.3]	[139.6]

UNIT	TRAVEL TIME
SHP08x20	0.1
SHP08x40	0.18
SHP12x20	0.18
SHP12x40	0.22
SHP 16x15	0.15
SHP16x35	0.2
SHP16x55	0.25

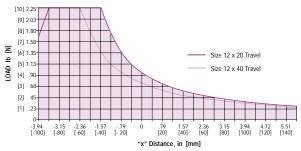
#### NOTES:

1) Travel time is in seconds from application of pressure. 2) Travel times relatively independent of pressure between 60 and 100 psi.



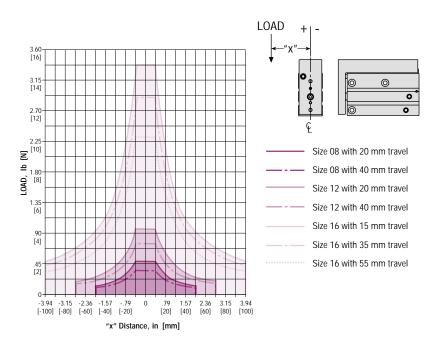
#### MAXIMUM DYNAMIC HORIZONTAL PITCH MOMENT LOADS

#### SIZE 12

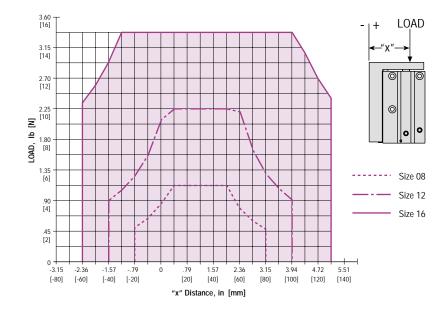




#### MAXIMUM DYNAMIC ROLL MOMENT LOADS



#### MAXIMUM DYNAMIC VERTICAL PITCH MOMENT LOADS





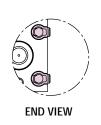
# **OPTION:** SERIES SHP SLIDES

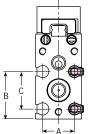


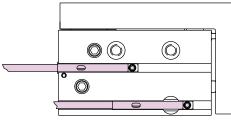
#### MAGNET FOR PHD SERIES 6790 MINIATURE REED AND SOLID STATE SWITCHES

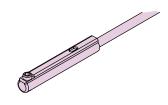
This option equips the unit with a magnetic piston for use with PHD's Series 6790 Switch. The switch housing is contained by the slide housing and provides a very compact switch design. The switches mount easily into two small grooves located on the side of the slide housing and are locked into place with a set screw.

PART NO.	DESCRIPTION
67902-1-02	NPN (Sink) or PNP (Source) DC Reed, 2 m cable
67902-1-05	NPN (Sink) or PNP (Source) DC Reed, 5 m cable
67903-1-02	NPN (Sink) DC Solid State, 2 m cable
67903-1-05	NPN (Sink) DC Solid State, 5 m cable
67904-1-02	PNP (Source) DC Solid State, 2 m cable
67904-1-05	PNP (Source) DC Solid State, 5 m cable
67922-1	NPN (Sink) or PNP (Source) DC Reed, Quick Connect
67923-1	NPN (Sink) DC Solid State, Quick Connect
67924-1	PNP (Source) DC Solid State, Quick Connect
63549-02	2 m Cordset with Quick Connect
63549-05	5 m Cordset with Quick Connect
67924-1 63549-02	PNP (Source) DC Solid State, Quick Connect 2 m Cordset with Quick Connect









<b>LETTER</b>	SIZI	E 08	SIZE	12	SIZE 16			
DIM	in	mm	in	mm	in	mm		
Α	0.433	11.0	0.636	16.2	0.786 0.892 0.447	20.0		
В	0.648	16.5	0.854	21.7	0.892	22.7		
С	0.510	13.0	0.446	11.3	0.447	11.4		

#### **REED BENEFITS**

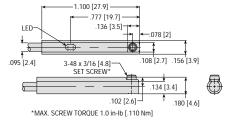
- Available as 4.5 30 VDC model for simple interfacing to sequencers and programmable controllers.
- Can be used to directly drive some types of relays or valve solenoids within the switch specifications stated.

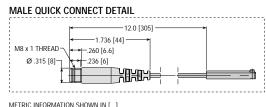
SPECIFICATIONS	67902 & 67922
OPERATING PRINCIPLE	Magnetic Reed
ACTUATED BY	Piston Magnet
INPUT VOLTAGE	4.5 - 30 VDC
OUTPUT TYPE	Contact Closure
CURRENT RATING	50 mA Max.
CONTACT RESISTANCE	.115 Ohm Max.
ENVIRONMENTAL	IP67
OPERATING TEMP.	-20° to 85°C

#### **SOLID STATE BENEFITS**

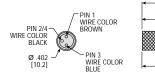
- Solid state switches afford long life. Constant amplitude output allows use with most digital logic systems.
  - Switch circuitry protects against voltage surges and other electrical anomalies associated with operating systems.
- Excellent switch hysteresis characteristics and symmetry.
- Offered in 4.5 30 VDC current sinking and current sourcing versions for simple interfacing to electronic system controllers.

SPECIFICATIONS	67903 & 67923	67904 & 67924					
OPERATING PRINCIPLE	Solid State						
ACTUATED BY	Piston	Magnet					
INPUT VOLTAGE	4.5 - 30 VDC						
OUTPUT TYPE	NPN (Sink)	PNP (Source)					
CURRENT RATING	50 m	A. Max					
VOLTAGE DROP	.5	VDC					
SWITCH BURDEN	10 m	A. Max.					
ENVIRONMENTAL	IF	P67					
OPERATING TEMP.	-20° t	to 85°C					





#### 63549-xx CORDSET WITH FEMALE QUICK CONNECT



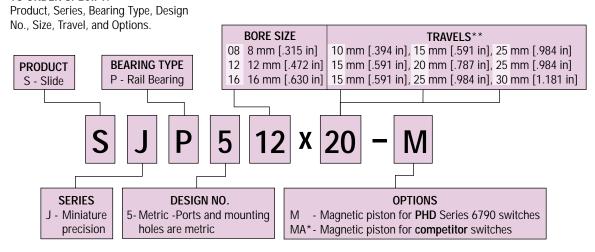
19.3] BLACK CABLE
-------------------

	LETTER DIM.
MODEL NO.	L
63549-02	78.74 [2 m]
63549-05	196.85 [5 m]



# **ORDERING DATA:** SERIES SJP RAIL BEARING SLIDES

#### TO ORDER SPECIFY:



PART NO.	DESCRIPTION
67902-1-02	NPN (Sink) or PNP (Source) DC Reed, 2 m cable
67902-1-05	NPN (Sink) or PNP (Source) DC Reed, 5 m cable
67903-1-02	NPN (Sink) DC Solid State, 2 m cable
67903-1-05	NPN (Sink) DC Solid State, 5 m cable
67904-1-02	PNP (Source) DC Solid State, 2 m cable
67904-1-05	PNP (Source) DC Solid State, 5 m cable
67922-1	NPN (Sink) or PNP (Source) DC Reed, Quick Connect
67923-1	NPN (Sink) DC Solid State, Quick Connect
67924-1	PNP (Source) DC Solid State, Quick Connect
63549-02	2 m Cordset with Quick Connect
63549-05	5 m Cordset with Quick Connect

#### NOTES:

- \*See page 5A-7 for use of the -MA option.
- \*\*Consult PHD for additional travel increments.

# **BENEFITS:** SERIES SJP RAIL BEARING SLIDES

#### **BENEFITS**

- This slide is designed to be an MRO drop-in. Consult PHD or your local distributor for unit compatibility.
- Series SJP Slides use rail bearing technology to provide smooth, precise movement with high accuracy within confined spaces.
- Available in three bore sizes with three travel lengths each.
- Standard internal shock pads eliminate metal to metal contact reducing noise and end-of-travel impact forces.
- Standard Series SJP Slides provide multiple mounting options for maximum flexibility. Bodies feature mounting holes on sides, end, and bottom.
- Slide body incorporates twin switch slots on both sides for convenient mounting of PHD's Series 6790 4 mm Reed and solid state switches. Magnet option -M is required when using Series 6790 Switches.
- Series SJP Slides can be fitted with certain competitor's switches. Magnet option -MA is required when using these switches.



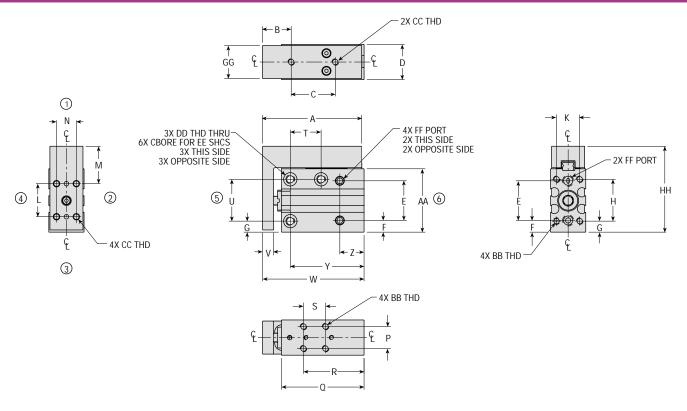
		08	1	2	16	
SPECIFICATIONS	in	mm	in	mm	in	mm
POWER SOURCE			Integral pneu	ımatic cylinde	er	
MAX. OPERATING PRESSURE			100 psi	[6.9 bar]		
OPERATING TEMPERATURE RANGE	-20° to 180°F [-29° to 82°C]					
TRAVELS	.394	10	.591	15	.591	15
	.591	15	.787	20	.984	25
	.984	25	.984	25	1.181	30
STANDARD FEATURES		Multiple port	positions, m	nultiple moun	ting position	S
BEARINGS	Stainless steel ground rail bearing system with recirculating ball bearing					
BODY	Anodized aluminum alloy					
TOOL PLATE			Anodized al	uminum alloy	1	

#### **FORCE TABLE**

	SIZI		SIZI		SIZE 16		
DIRECTION	lb/psi	N/bar	lb/psi	N/bar	lb/psi	N/bar	
EXTEND	.079	5.1	.175	11.3	.314	20.3	
RETRACT	.060	3.9	.132	8.5	.270	17.4	



# **DIMENSIONS:** SERIES SJP RAIL BEARING SLIDES



LETTER		т,	SIZE					т.	SIZE 12 TRAVEL, in [mm]				SIZE 16 TRAVEL, in [mm]					
DIM.	.394	[10]	.591	<u>in (mm)</u> [15]	.984	[25]	.591	[15]	.787	<u>ın (mm)</u> [20]	.984	[25]	.591	[15]	.984	<u>in [mm</u> [25]	1.181	[30]
A	1.969	[50.0]	2.165	[55.0]	2.559	[65.0]	2.441	[62.0]	2.638	[67.0]	2.953	[75.0]	2.953	[75.0]	3.346	[85.0]	3.543	[90.0]
B	1.707	[50.0]	0.512	[13.0]	2.557	[03.0]	2.441	[02.0]	0.591	[15.0]	2.755	[73.0]	2.733	[75.0]	0.709	[18.0]	3.343	[70.0]
C	.787	[20.0]	0.984	[25.0]	1.575	[40.0]	.984	[25.0]	1.181	[30.0]	1.575	[40.0]	1.378	[35.0]	1.772	[45.0]	1.969	[50.0]
D	.,,,,	[20.0]	.630	[16.0]	1.070	[ 10.0]	.701	[20.0]	.787	[20.0]	1.070	[10.0]	1.070	[00.0]	.984	[25.0]	1.707	[00:0]
E			.709	[18.0]					.866	[22.0]					.984	[25.0]		
F			.209	[5.3]					.217	[5.5]					.256	[6.5]		
G			.197	[5.0]					.197	[5.0]					.217	[5.5]		
Н			.748	[19.0]					.906	[23.0]					1.063	[27.0]		
K			.413	[10.5]					.512	[13.0]					.669	[17.0]		
L			.591	[15.0]					.709	[18.0]					1.024	[26.0]		
M			.669	[17.0]					.846	[21.5]					.846	[21.5]		
N			.354	[9.0]					.433	[11.0]					.630	[16.0]		
Р			.394	[10.0]					.512	[13.0]					.669	[17.0]		
Q	1.673	[42.5]	1.870	[47.5]	2.264	[57.5]	2.028	[51.5]	2.224	[56.5]	2.539	[64.5]	2.441	[62.0]	2.835	[72.0]	3.031	[77.0]
R	1.279	[32.5]	1.476	[37.5]	1.870	[47.5]	1.555	[39.5]	1.752	[44.5]	2.067	[52.5]	1.969	[50.0]	2.362	[60.0]	2.559	[65.0]
S	.591	[15.0]	.787	[20.0]	1.181	[30.0]	.709	[18.0]	.945	[24.0]	1.260	[32.0]	1.181	[30.0]	1.575	[40.0]	1.772	[45.0]
T	.551	[14.0]	.945	[24.0]	1.339	[34.0]	.945	[24.0]	.945	[24.0]	1.339	[34.0]	1.339	[34.0]	1.575	[40.0]	1.575	[40.0]
U			.748	[19.0]					.906	[23.0]					1.063	[27.0]		
V			.197	[5.0]					.276	[7.0]					.374	[9.5]		
W	2.008	[51.0]	2.205	[56.0]	2.598	[66.0]	2.480	[63.0]	2.677	[68.0]	2.992	[76.0]	2.993	[76.0]	3.386	[86.0]	3.583	[91.0]
Υ	1.516	[38.5]	1.713	[43.5]	2.107	[53.5]	1.831	[46.5]	2.028	[51.5]	2.343	[59.5]	2.244	[57.0]	2.638	[67.0]	2.835	[72.0]
Z			.433	[11.0]					.492	[12.5]					.512	[13.0]		
AA			1.132	[28.8]					1.319	[33.5]					1.516	[38.5]		
BB			[M3 x 0	.5 x 4.8]					[M4 x 0	.7 x 6.0]					[M4 x (	0.7 x 6.0	)]	
CC			[M3 x 0	0.5 x 5]					[M4 x 0.7	x 6 MIN	]				[M4 x 0.	7 x 6 M	N]	
DD	[M4 x 0.7]							[M5:	x 0.8]					[M5	x 0.8]			
EE	[M3]				[M4]							[]	M4]					
FF			[M5 x 0.	8 PORT]					[M5 x 0.	8 PORT]					[M5 x 0	).8 POR	Γ]	
GG			.591	[15.0]					.748	[19.0]					.945	[24.0]		
HH			1.536	[39.0]					1.850	[47.0]					2.106	[53.5]		

#### NOTES:

- NOTES:

  1) DESIGNATED & IS CENTERLINE OF UNIT.

  2) METRIC INFORMATION SHOWN IN [].

  3) CIRCLED NUMBERS INDICATE POSITION CALLOUT.



# ENGINEERING DATA: SERIES SJP RAIL BEARING SLIDES

#### PRESSURE RATINGS

All Series SJP Slides have an operating pressure range of 20 psi minimum to 100 psi maximum [1.4 to 6.9 bar]. Maximum life will be achieved when pressure and velocity are no greater than necessary for proper operation. External flow controls are recommended. Series SJP Slides feature standard pneumatic ports on the end and both sides of the slide body, and are provided with the end ports ready for use and the side ports plugged with set screws and thread sealant.

#### OPERATING TEMPERATURE

Series SJP Slides are designed for use in temperatures between -20 $^{\circ}$  to 180 $^{\circ}$  F [-29 $^{\circ}$  to 82 $^{\circ}$  C]. For temperatures outside this range, consult PHD.

#### **SEALS**

Series SJP Slides utilize urethane and Nitrile seals which are compatible with standard paraffin-based lubrication oils used for pneumatic cylinders. For compatibility with other fluids, consult PHD.

#### LUBRICATION

All units are pre-lubricated at the factory for service under normal operating conditions. Slides are designed and tested with non-lubricated air. However, the use of lubricated air will extend life.

#### **SLIDE SELECTION**

There are three major factors to consider when selecting a slide: thrust capacity, dynamic moment capacity, and the allowable velocity.

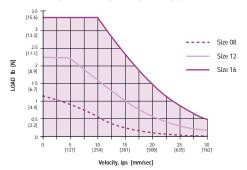
#### 1 THRUST CAPACITY

Use the theoretical force output table to determine if thrust is sufficient for the applied load.

#### 2 DYNAMIC MOMENT CAPACITY

The Dynamic Moment Load graphs show the allowable load for the three most common mounting positions of the Series SJP Slide. Determine the distance "x" from the edge of the tool plate to the load

#### ALLOWABLE LOAD VS. VELOCITY



#### TOTAL TRAVEL LENGTH AND WEIGHT

Tolerance of minimum travel length is +.039/-.000 [+1 mm/-0 mm].

	MININ		UNIT BASE WEIGHT				
SIZE	in	mm	Ib	kg			
	(.394)	10	.16	.07			
80	(.591)	15	.18	.08			
	(.984)	25	.21	.10			
	(.591)	15	.32	.15			
12	(.787)	20	.35	.16			
	(.984)	25	.39	.18			
	(.591)	15	.57	.26			
16	(.984)	25	.65	.29			
	(1.181)	30	.68	.31			

#### MATERIAL SPECIFICATIONS

The slide housing and tool plate are anodized aluminum alloy. Linear rail and bearings are hardened and ground stainless steel.

#### **MAINTENANCE**

In common with most PHD products, these slides are fully field repairable. Repair kits and main structural components are available as needed for extended service life.

#### MOUNTING INSTRUCTIONS

PHD recommends to mount load or tooling with tool plate retracted. Support tool plate while tightening fasteners. Maximum mounting torques (for screw thread engagement of one diameter or greater):

<b>FASTENER SIZE</b>	in-lb	Nm
M3	9	1
M4	18	2
M5*	35	4

<sup>\*</sup>Mounting holes only, port fittings require less.

#### THEORETICAL FORCE OUTPUT TABLE | Ib [N]

		OPERATING PRESSURE								
		20 psi	30 psi	40 psi	50 psi	60 psi	70 psi	80 psi	90 psi	100 psi
SIZE	DIRECTION	[1.4 bar]	[2.1 bar]	[2.8 bar]	[3.4 bar]	[4.1 bar]	[4.8 bar]	[5.5 bar]	[6.2 bar]	[6.9 bar]
	RFTRACT	1.2	1.8	2.4	3.0	3.6	4.2	4.8	5.4	6.0
08	KEIKACI	[5.3]	[8.0]	[10.7]	[13.3]	[16.0]	[18.7]	[21.3]	[24.0]	[26.7]
06	EXTEND	1.6	2.4	3.1	3.9	4.7	5.5	6.3	7.1	7.9
		[7.1]	[10.7]	[13.8]	[17.3]	[20.9]	[24.4]	[28.0]	[31.6]	[35.1]
	RETRACT	2.7	4.0	5.3	6.7	8.0	9.3	10.7	12.0	13.3
12		[12.0]	[17.8]	[23.6]	[29.8]	[35.6]	[41.3]	[47.6]	[53.3]	[59.1]
12	EXTEND	3.5	5.3	7.1	8.8	10.6	12.4	14.1	15.9	17.6
		[15.6]	[23.6]	[31.6]	[39.1]	[47.1]	[55.1]	[62.7]	[70.7]	[78.2]
	RETRACT	5.4	8.1	10.8	13.5	16.2	18.9	21.6	24.3	27.0
16	KLIKACI	[24.0]	[36.0]	[48.0]	[60.0]	[72.0]	[84.0]	[96.0]	[108.0]	[120.0]
10	FXTFND	6.3	9.4	12.5	15.7	18.8	22.0	25.1	28.2	31.4
	EXTEND	[28.0]	[41.8]	[55.6]	[69.8]	[83.6]	[97.8]	[111.6]	[125.3]	[139.6]

center of gravity. Use the appropriate graph for the loading condition to determine the allowable load. It is generally best to keep the center of gravity of the load as close to the slide as possible. (See next page for graphs.) If the application requires combined loading such as a horizontal pitch load combined with a roll load, if static loads exceed dynamic loads, or if there are other questions concerning the selection of an appropriate slide, please contact PHD's Customer Service Department.

#### 3

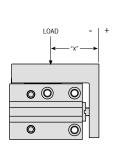
#### ALLOWABLE VELOCITY

Use the allowable velocity graph at left to verify that the slide selected can carry the payload at the desired velocity.



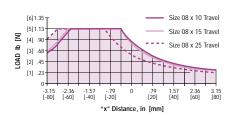
# **ENGINEERING DATA:** SERIES SJP RAIL BEARING SLIDES

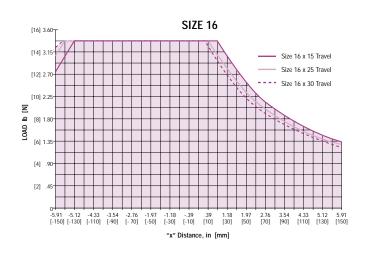
#### MAXIMUM DYNAMIC HORIZONTAL PITCH MOMENT LOADS



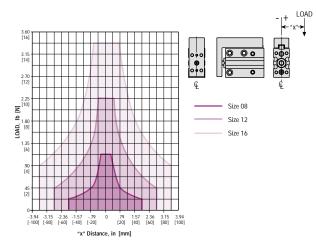


SIZE 08

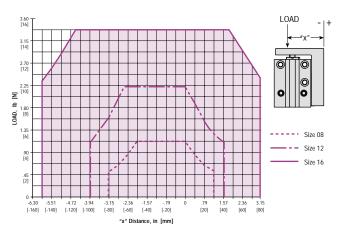




#### MAXIMUM DYNAMIC ROLL MOMENT LOADS



#### MAXIMUM DYNAMIC VERTICAL PITCH MOMENT LOADS



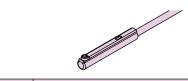


# **OPTIONS:** SERIES SJP RAIL BEARING SLIDES

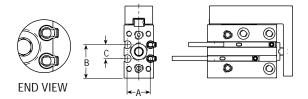


#### MAGNET FOR PHD SERIES 6790 REED AND SOLID STATE SWITCHES

This option equips the unit with a magnetic piston for use with PHD's Series 6790 Switch. The switch housing is contained by the slide housing and provides a very compact switch design. The switches mount easily into two small grooves located on the side of the slide housing and are locked into place with a set screw.



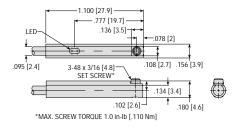
	LETTER	SIZI	80 =	SIZE	12	SIZE 16		
	DIM	in mm		in mm		in	mm	
Ī	Α	0.496	12.6	0.645	16.4	0.786 0.972 0.447	20.0	
	В	0.712	18.1	0.844	21.4	0.972	24.7	
	С	0.297	7.5	0.388	9.9	0.447	11.4	



#### **REED BENEFITS**

- Available as 4.5 30 VDC model for simple interfacing to sequencers and programmable controllers.
- Can be used to directly drive some types of relays or valve solenoids within the switch specifications stated.

SPECIFICATIONS	67902 & 67922			
OPERATING PRINCIPLE	Magnetic Reed			
ACTUATED BY	Piston Magnet			
INPUT VOLTAGE	4.5 - 30 VDC			
OUTPUT TYPE	Contact Closure			
CURRENT RATING	50 mA Max.			
CONTACT RESISTANCE	.115 Ohm Max.			
ENVIRONMENTAL	IP67			
OPERATING TEMP.	-20° to 85°C			





# MAGNET FOR COMPETITOR'S REED AND SOLID STATE SWITCHES

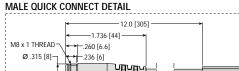
The Series SJP can be used as an exact drop-in replacement for a competitor's commercially available rail slide. If the Series SJP is being used to replace the competitor's slide, and you wish to continue using their switches, the MA option must be specified. For additional information and switch compatibility, contact your local PHD distributor or PHD customer service.

PART NO.	DESCRIPTION
67902-1-02	NPN (Sink) or PNP (Source) DC Reed, 2 m cable
67902-1-05	NPN (Sink) or PNP (Source) DC Reed, 5 m cable
67903-1-02	NPN (Sink) DC Solid State, 2 m cable
67903-1-05	NPN (Sink) DC Solid State, 5 m cable
67904-1-02	PNP (Source) DC Solid State, 2 m cable
67904-1-05	PNP (Source) DC Solid State, 5 m cable
67922-1	NPN (Sink) or PNP (Source) DC Reed, Quick Connect
67923-1	NPN (Sink) DC Solid State, Quick Connect
67924-1	PNP (Source) DC Solid State, Quick Connect
63549-02	2 m Cordset with Quick Connect
63549-05	5 m Cordset with Quick Connect

#### **SOLID STATE BENEFITS**

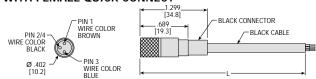
- Solid state switches afford long life. Constant amplitude output allows use with most digital logic systems.
- Switch circuitry protects against voltage surges and other electrical anomalies associated with operating systems.
- Excellent switch hysteresis characteristics and symmetry.
- Offered in 4.5 30 VDC current sinking and current sourcing versions for simple interfacing to electronic system controllers.

SPECIFICATIONS	67903 & 67923	67904 & 67924			
OPERATING PRINCIPLE	Solid	State			
ACTUATED BY	Piston	Magnet			
INPUT VOLTAGE	4.5 - 3	30 VDC			
OUTPUT TYPE	NPN (Sink)	PNP (Source)			
CURRENT RATING	50 mA. Max				
VOLTAGE DROP	.5 VDC				
SWITCH BURDEN	10 mA. Max.				
ENVIRONMENTAL	IP67				
OPERATING TEMP.	-20° to 85°C				



METRIC INFORMATION SHOWN IN [ ]

#### 63549-xx CORDSET WITH FEMALE QUICK CONNECT

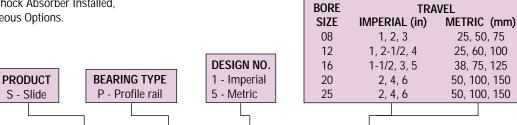


	LETTER DIM.
MODEL NO.	L
63549-02	78.74 [2 m]
63549-05	196.85 [5 m]



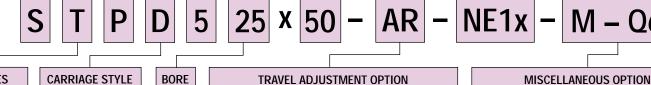
#### TO ORDER SPECIFY:

Product, Series, Bearing Type, Carriage Style, Design No., Size, Travel, Travel Adjustment, Shock Absorber Installed, and Miscellaneous Options.



#### SHOCK ABSORBER INSTALLED OPTION

- NE1x Shock absorber installed on extend, position 1
- NE2x Shock absorber installed on extend, position 2
- NRx Shock absorber installed on retract
- "x" indicates shock absorber dampening constant (2 or 3) which must be specified by the customer.



#### **SERIES** T - Compact Thruster

#### SIZE 80 12 16 20

25

#### TRAVEL ADJUSTMENT OPTION

- AE1 Travel adjustment and shock pad on extend, position 1
- AE2 Travel adjustment and shock pad on extend, position 2
- AR Travel adjustment and shock pad on retract

#### MISCELLANEOUS OPTIONS

ORDERING DATA: SERIES STP SLIDES WITH RAIL BEARING

- J3 Transitional fit dowel holes
- J8 Precision fit dowel holes
- M Magnetic piston for use with PHD Series 6790 Switches
- Q6 Complete corrosion resistant guide shafts

#### **SERIES 6790 SWITCHES**

D - Dual stop

PART NO.	DESCRIPTION
67902-1-02	NPN (Sink) or PNP (Source) DC Reed, 2 m cable
67902-1-05	NPN (Sink) or PNP (Source) DC Reed, 5 m cable
67903-1-02	NPN (Sink) DC Solid State, 2 m cable
67903-1-05	NPN (Sink) DC Solid State, 5 m cable
67904-1-02	PNP (Source) DC Solid State, 2 m cable
67904-1-05	PNP (Source) DC Solid State, 5 m cable
67922-1	NPN (Sink) or PNP (Source) DC Reed, Quick Connect
67923-1	NPN (Sink) DC Solid State, Quick Connect
67924-1	PNP (Source) DC Solid State, Quick Connect
63549-02	2 meter Cordset with Quick Connect
63549-05	5 meter Cordset with Quick Connect

#### SHOCK ABSORBER REPLACEMENTS

<b>BORE SIZE</b>	PART NO.
08	68149-01-x
12	68149-01-x
16	68015-01-x
20	70861-01-x
25	67127-01-x



# **BENEFITS:** SERIES STP SLIDES WITH RAIL BEARING

#### **BENEFITS**

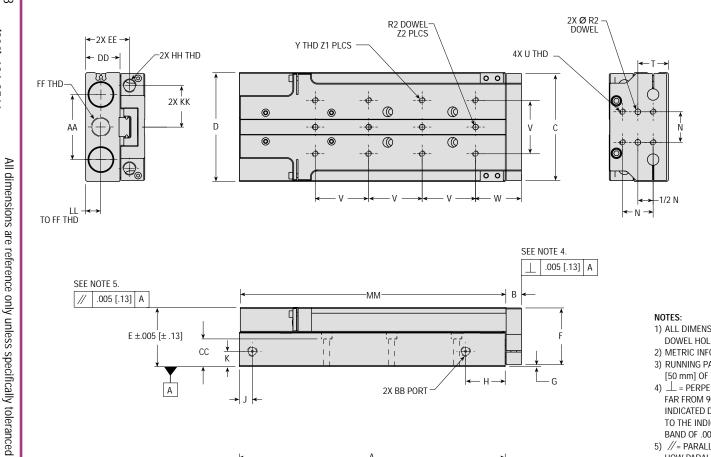
- Series STP Slides provide smooth precise linear motion with high accuracy at twice the thrust of a single bore cylinder.
- Available in five bore sizes with a choice of three travel lengths to fit a wide range of applications. Optional travel adjustments allow precise adjustment of travel.
- Available in both imperial and metric versions for applications in worldwide markets.
- Standard internal and external shock pads eliminate metal to metal contact reducing noise and end-of-travel impact forces.
- Fully integrated shock absorbers and travel adjustments with shock pads allow easy adjustment from back of slide while not increasing overall package width.
- Shock absorbers and travel adjustments on retract eliminate pitch and yaw moments usually associated with externally mounted shocks and travel adjustments.
- Series STP units have increased stopping capacity and minimal deflection when travel adjustment and/or shock absorbers are used in both positions.
- Slide housing incorporates twin switch slots for flush installation of PHD's new Series 6790 reed and solid state switches.
- Consult PHD for 4 mm and 8 mm proximity switch ready units.
- All units have customer mounting and dowel holes in the housing, carriage, and tool plate.
- Modular mounting kits allow quick connect of same size Series STP Slides without the need for transition plates. See page 5A-34 for details.
- Standard stainless steel fasteners eliminate the need for costly corrosion resistant options. Combining this with -Q6 shaft option provides a completely corrosion resistant unit.

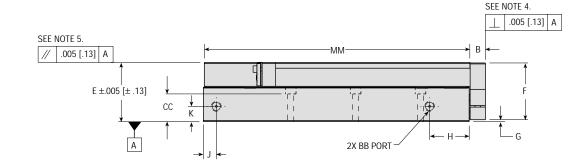


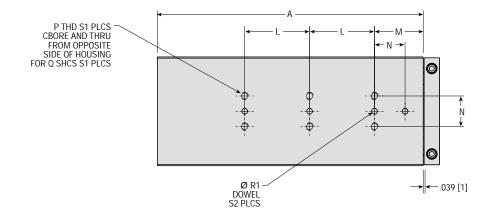


	BORE SIZE						
SPECIFICATIONS	08	12, 16, 20, & 25					
POWER SOURCE	Two cylinde	ers built into the Slide Body					
WORKING PRESSURE	20 psi min	20 psi min - 150 psi max at zero load					
TEMPERATURE RANGE	-20° to +180°F [-29° to 82°C]						
LUBRICATION	Permanent for N	Ion-Lubricated or Lubricated Air					
GUIDE SHAFTS	Stainless Steel	Hard Chrome Plated Steel					
BEARINGS	Stainless Steel Ground Ra	il System with Recirculating Ball Bearings					
CARRIAGE	Anodized Aluminum Alloy						
TOOL PLATE	Anodized Aluminum Alloy						
BODY	Anoc	lized Aluminum Alloy					









#### NOTES:

- 1) ALL DIMENSIONS ARE SYMMETRICAL ABOUT CENTERLINE OF DOWEL HOLES UNLESS OTHERWISE SPECIFIED.
- 2) METRIC INFORMATION SHOWN IN [ ].
- 3) RUNNING PARALLELISM TO DATUM A IS .002 in [.05 mm] AT 2 in [50 mm] OF TRAVEL.
- 4)  $\perp$  = PERPENDICULARITY TOLERANCE, THIS DETERMINES HOW FAR FROM 90° THAT THE INDICATED FEATURES CAN BE TO THE INDICATED DATUM FEATURES. THIS SURFACE IS ORIENTED (90°) TO THE INDICATED DATUM SURFACES WITHIN A TOLERANCE BAND OF .005 [.13].
- 5) //= PARALLELISM TOLERANCE, THIS TOLERANCE DETERMINES HOW PARALLEL (180°) THAT THE INDICATED FEATURES CAN BE TO THE INDICATED DATUM FEATURES. THE SURFACE IS PARALLEL (180°) TO THE INDICATED DATUM SURFACES WITHIN A TOLERANCE BAND OF .005 [.13].

All dimensions are reference only unless specifically toleranced

#### **BORE SIZE LETTER** 20 08 12 25 16 DIM. in mm mm mm mm in mm in mm in mm in mm mm mm mm in mm in mm mm 2.5 5 100 100 TRAVEL 1-1/2 75 125 150 2.953 75.0 4.587 116.5 5.866 149.0 3.544 90.0 5.671 144.0 7.795 198.0 4.528 115.0 6.024 153.0 8.544 217.0 5.433 138.0 8.327 211.5 10.965 278.5 5.531 140.5 8.327 211.5 10.965 278.5 Α В 0.315 8.0 0.394 10.0 0.433 11.0 0.512 13.0 0.630 16.0 2.087 53.0 2.559 65.0 3.425 87.0 3.7 94.0 4.409 112.0 С D 2.165 55.0 2.638 67.0 3.504 89.0 3.779 96.0 4.488 114.0 Ε 0.983 25.0 1.37 34.8 1.574 40.0 1.969 50.0 2.440 62.0 0.924 23.5 1.271 32.3 1.476 37.5 1.87 47.5 2.341 59.5 0.079 G 0.039 1.0 0.079 2.0 0.079 2.0 2.0 0.079 2.0 Н 23.5 0.797 1.437 1.2 30.5 41.5 0.924 20.2 36.5 1.634 0.492 12.5 0.797 20.2 0.531 13.5 1.2 30.5 0.532 13.5 5.5 11.5 Κ 0.217 0.378 9.6 0.453 0.566 14.4 0.630 16.0 1.102 28.0 1.496 38.0 1.929 49.0 2.205 56.0 2.677 68.0 1.142 29.0 1.024 26.0 1.713 43.5 1.613 41.0 2.008 51.0 M Ν 0.551 14.0 0.67 17.0 0.905 23.0 1.024 26.0 1.260 32.0 Ρ 8-32 x .394 [M4 x 0.7 x 10] 5/16-18 x 1.142 [M8 x 1.25 x 29] 10-24 x .632 [M5 x 0.8 x 16] 1/4-20 x .689 [M6 x 1.0 x 17.5] 1/4-20 x .910 [M6 x 1.0 x 23] Q #5 [M3] #6 [M4] #10 [M5] #10 [M5] 1/4 [M6] 6 mm x 5.5 mm DP R1 3 mm x 2.5 mm DP 4 mm x 2.5 mm DP 5 mm x 3 mm DP 5 mm x 3 mm DP R2 3 mm x 3 mm DP 4 mm x 4 mm DP 5 mm x 5 mm DP 5 mm x 5 mm DP 6 mm x 6 mm DP S1 6 4 4 6 4 4 6 4 6 4 4 6 S2 4 0.5315 13.5 0.713 18.1 0.9055 23.0 1.078 27.4 1.260 32.0 Τ 5-40 x .315 DP [M3 x 8 DP] 10-24 x .433 DP [M5 x 11 DP] 10-24 x .512 DP [M5 x 13 DP] 1/4-20 x .630 DP [M6 x 16 DP] U 6-32 x .394 DP [M4 x 10 DP] ٧ 0.868 22.0 1.102 28.0 1.496 38.0 1.654 42.0 2.205 56.0 28.0 1.300 33.0 35.0 1.85 47.0 48.0 W 1.102 1.378 1.890 Υ 5-40 x .275 DP [M3 x 7 DP] 8-32 x .375 DP [M4 x 9.5 DP] 10-24 x .375 DP [M5 x 9.5 DP] 10-24 x .375 DP [M5 x 9.5 DP] 1/4-20 x .500 DP [M6 x 13 DP] Z1 4 3 4 4 2 3 4 Z2 2 2 29.0 38.0 2.264 57.5 68.2 1.142 1.496 1.969 50.0 2.684 AA 10-32 PORT [M5 x 0.8 PORT] 10-32 PORT [M5 x 0.8 PORT] 10-32 PORT [M5 x 0.8 PORT] 1/8 NPT PORT [1/8 BSPP PORT] 1/8 NPT PORT [1/8 BSPP PORT] BB 0.401 10.2 0.612 15.5 0.684 17.4 0.91 23.1 1.143 29.0 CC 0.795 20.2 23.0 1.132 28.8 1.418 DD 0.551 14.0 0.906 36.0 EE 0.752 19.1 1.022 26.0 1.181 30.0 1.447 36.8 1.810 46.0 7/16-20 x 1.800 DP FF M6 x 1.0 x 32 DP M14 x 1.5 x 55 DP M16 x 1.5 x 57 DP M20 x 1.5 x 60 DP НН M8 x 1.0 x 12.5 DP M8 x 1.0 x 12.5 DP M10 x 1.0 x 18.5 DP M12 x 1.0 x 19 DP M14 x 1.5 x 19.5 DP 0.955 1.712 KK 0.743 18.9 24.3 1.26 32.0 1.417 36.0 43.5 LL 5.5 0.319 8.1 0.394 0.217 10.0 0.566 14.4 0.620 15.7 2.953 75.0 | 4.587 116.5 | 5.866 149.0 | 3.377 85.8 5.671 144.0 7.795 198.0 | 4.528 115.0 | 6.024 153.0 | 8.544 217.0 | 5.433 138.0 | 8.327 211.5 | 10.965 278.5 | 5.531 140.5 | 8.327 211.5 | 10.965 278.5

DIMENSIONS: SERIES STP SLIDES WITH RAIL BEARING

METRIC INFORMATION SHOWN IN [ ]

#### PRESSURE RATINGS

All Series STP Slides have an operating pressure range of 20 psi minimum to 150 psi maximum [1.4 to 10 bar].

#### **BREAKAWAY**

Units have less than 20 psi [1.4 bar] breakaway with zero load.

#### **OPERATING TEMPERATURE**

Series STP Slides are designed for use in temperatures between 20° to 180° F [-6° to 82° C]. For temperatures outside this range, consult PHD.

#### **SEALS**

Series STP Slides utilize urethane and Nitrile seals which are compatible with standard paraffin-based lubrication oils used for pneumatic cylinders. For compatibility with other fluids, consult PHD.

#### LUBRICATION

All units are pre-lubricated at the factory for service under normal operating conditions. Slides are designed and tested with non-lubricated air. However, the use of lubricated air will extend life.

#### OPERATING SLIDE VELOCITY

For sizes 08 and 12, slide velocity is 36 in/sec [914 mm/sec] for extend and 24 in/sec [610 mm/sec] on retract. For sizes 16, 20, and 25, slide velocity is 30 in/sec [962 mm/sec] for extend and 24 in/sec [610 mm/sec] on retract. These values are based on an unloaded slide at 87 psi [6 bar] operating pressure.

#### MATERIAL SPECIFICATIONS

The slide housing, tool plate, and carriage are anodized aluminum alloy. Linear rails and bearings are hardened and ground stainless steel. On standard sizes 12, 16, 20, and 25 slides, the shafts are hard chrome plated steel. Size 08 slides have stainless steel shafts. The corrosion resistant option on sizes 12, 16, 20, and 25 provides hard chrome plated stainless steel shafts.

#### **MAINTENANCE**

In common with most PHD products, these slides are fully field repairable. Repair kits and main structural components are available as needed for extended service life.

#### TOTAL TRAVEL LENGTH

Tolerance on specified minimum travel length is +.098/-.000 [+2.5mm/-0mm].

	MINIMUM TRAVEL							
SIZE	in mm							
	1	25						
80	2	51						
	3	76						
	1	25						
12	2.5	64						
	4	102						
	1-1/2	38						
16	3	76						
	5	127						
	2	51						
20	4	102						
	6	152						
	2	51						
25	4	102						
	6	152						

#### **UNIT WEIGHT**

					OPTION ADDERS						
	TRA	VEL	BASE WEIGHT		-AR		-NRx		-AEx O	-AEx OR NEx	
SIZE	in	mm	lb	kg	lb	kg	lb	kg	lb	kg	
	1	25	0.55	0.25	0.03	0.014	0.11	0.05			
80	2	50	0.81	0.37	0.04	0.018	0.11	0.05	0.06	0.03	
	3	75	1.01	0.46	0.05	0.023	0.11	0.05			
	1	25	1.12	0.51	0.10	0.05	0.09	0.04			
12	2-1/2	60	1.71	0.78	0.15	0.07	0.178	0.08	0.09	0.04	
	4	100	2.26	1.03	0.20	0.09	0.298	0.14			
	1-1/2	38	2.10	0.95	0.22	0.10	0.19	0.09			
16	3	75	2.68	1.22	0.29	0.13	0.26	0.12	0.13	0.06	
	5	125	3.63	1.65	0.40	0.18	0.37	0.17			
	2	50	3.62	1.64	0.65	0.30	0.32	0.15			
20	4	100	5.24	2.38	0.85	0.39	0.512	0.23	0.27	0.12	
	6	150	6.64	3.01	1.03	0.47	0.687	0.31			
	2	50	5.46	2.48	0.57	0.26	0.42	0.19			
25	4	100	7.55	3.43	0.87	0.39	0.73	0.33	0.29	0.13	
	6	150	9.55	4.34	1.16	0.53	1.02	0.46			



#### SLIDE SELECTION

There are three major factors to consider when selecting a slide: thrust capacity, allowable static and dynamic moment capacity, and table deflection (as either pitch, yaw, or roll).

#### 1 THRUST CAPACITY

Use the effective piston area (see thrust specifications) of the slide to determine if thrust is sufficient for the applied load.

# 2 STATIC AND DYNAMIC MOMENT CAPACITY The maximum static moments for all units are listed in the

static moments for all units are listed in the static moment chart below and must not be exceeded. The maximum allowable dynamic moment is equal to 1/10 the maximum static moment in consideration of the load inertia. Calculate static and dynamic moments of the system using the following equations and diagrams:

Mp (Pitch) =  $(Ah + CG) \times LOAD$  or  $(Av + CG) \times LOAD$ My  $(Yaw) = (Ah + CG) \times LOAD$  or  $CG \times LOAD$ Mr  $(Roll) = (Av + CG) \times LOAD$  or  $CG \times LOAD$ 

(continued on next six pages)

#### THRUST SPECIFICATIONS

	SHAFT DIAMETER		BORE DIAMETER		SHAFT	EFFECTIVE PISTON AREA	
SIZE	in	mm	in mm		DIRECTION	in <sup>2</sup>	mm²
08	0.157	4	0.315	8	EXTEND	0.16	101
					RETRACT	0.12	75
12	0.236	6	0.475	12	EXTEND	0.35	229
					RETRACT	0.27	172
16	0.315	8	0.630	16	EXTEND	0.62	402
					RETRACT	0.47	302
20	0.394	10	0.787	20	EXTEND	0.97	628
					RETRACT	0.73	470
25	0.472	12	0.984 25		EXTEND	1.52	982
					RETRACT	1.17	756

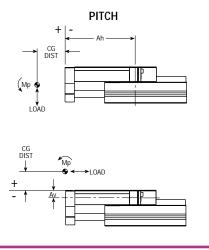
# CYLINDER THRUST CALCULATION IMPERIAL METRIC

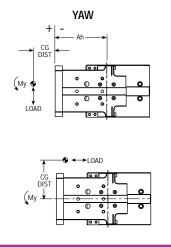
	IIVII LIXIAL	IVILITATE
	$F = P \times A$	$F = 0.1 \times P \times A$
F = Cylinder Force	lb	N
P = Operating Pressure	psi	bar
A = Effective Area	in²	$mm^2$

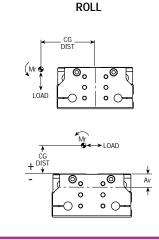
#### STATIC MOMENT CHART

	TRA	VEL	MAX PITCH MOMENT (Mp)		MAX YAW MOMENT (My)		MAX ROLL MOMENT (Mr)		MOMENT ARM Ah		MOMENT ARM Av	
SIZE	in	mm	in-lb	Nm	in-lb	Nm	in-lb	Nm	in	mm	in	mm
	1	25	42.4	4.8	42.4	4.8	67	7.6	2.442	62.0		
08	2	50	168	19.0	141	15.9	76	8.6	3.830	97.3	0.335	8.5
	3	75	227	25.6	190	21.5	76	8.6	4.914	124.8		
	1	25	146	16.5	124	14.0	127	14.4	2.717	69.0		
12	2-1/2	60	351	39.7	298	33.7	181	20.5	4.557	115.7	0.453	11.5
	4	100	474	53.6	403	45.5	181	20.5	6.308	160.2		
	1-1/2	38	238	26.9	200	22.6	271	30.6	3.711	94.3		
16	3	75	488	55.1	410	46.3	271	30.6	5.049	128.2	0.492	12.5
	5	125	664	75.0	558	63.0	271	30.6	7.292	185.2		
	2	50	497	56.2	418	47.2	550	62.2	4.286	108.9		
20	4	100	1290	145.8	1084	122.5	733	82.9	6.721	170.7	0.61	15.5
	6	150	1772	200.2	1488	168.1	733	82.9	9.034	229.5		
	2	50	796	89.9	668	75.5	991	112	4.488	114.0		
25	4	100	1592	179.9	1338	151.2	991	112	6.811	173.0	0.748	19.0
	6	150	2112	238.6	1774	200.4	991	112	9.194	233.5		

For more detail in determining table deflection; see page 5A-22 for pitch, page 5A-24 for yaw, and page 5A-26 for roll.



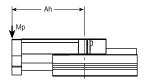






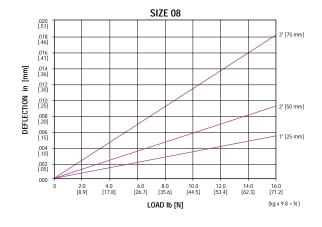
#### STATIC DEFLECTIONS IN PITCH

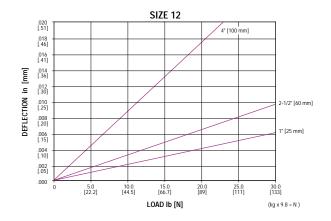
The graphs on this page show table pitch deflection due to static moment loads applied at distance Ah from bearing center while the unit is extended.

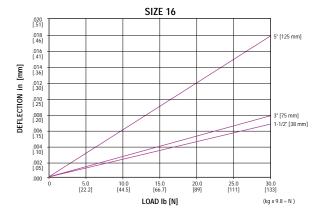


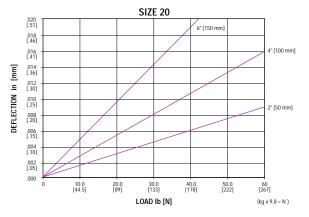
			MOMEN	IT ARM		
	TRA	VEL	Ah			
SIZE	in mm		in	mm		
	1	25	2.442	62.0		
80	2	50	3.830	97.3		
	3	75	4.914	124.8		
	1	25	2.717	69.0		
12	2-1/2	60	4.557	115.7		
	4	100	6.308	160.2		
	1-1/2	38	3.711	94.3		
16	3	75	5.049	128.2		
	5	125	7.292	185.2		
	2	50	4.286	108.9		
20	4	100	6.721	170.7		
	6	150	9.034	229.5		
	2	50	4.488	114.0		
25	4	100	6.811	173.0		
	6	150	9.194	233.5		

All tabulated and plotted values are typical and were determined empirically.













#### IMPERIAL EXAMPLE:

Determine the pitch deflection of a STPD125 x 6 slide at the center of gravity (CG) of a 10 lb load weight attached to the tool plate. The CG of the load is 2" further from the tool plate.

Calculate the moment of the application and the equivalent load at distance Ah.

Mp = Load x (Ah distance + CG distance)  
= 
$$10 \times (9.194 + 2) = 112 \text{ in-lb}$$

Equivalent load = (Mp / Ah) = 112 / 9.194 = 12 lb

Read the graph for a 12 lb load, deflection is approximately .003".

Deflection Ratio = Deflection at tool plate / Ah distance = .003 / 9.194 = 3.26 x 10<sup>-4</sup>

Deflection at load = Deflection Ratio x (Ah + CG)  
= 
$$3.26 \times 10^{-4} \text{ x } (9.194 + 2) = .0037^{\text{m}}$$

#### **METRIC EXAMPLE:**

Determine the pitch deflection of a STPD525 x 150 slide at the center of gravity (CG) of a 45 N load weight attached to the tool plate. The CG of the load is 50 mm further from the tool plate.

Calculate the moment of the application and the equivalent load at distance Ah.

Equivalent load = 
$$(Mp / Ah) \times 1000 = 12.76 / 233.5 \times 1000 = 55 N$$

Read the graph for a 55 N load, deflection is approximately .08 mm.

Deflection Ratio = Deflection at tool plate / Ah distance = 
$$.08 / 233.5 = 3.4 \times 10^{-4}$$

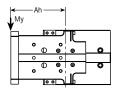
Deflection at load = Deflection Ratio x (Ah + CG)  
= 
$$3.4 \times 10^{-4} \times (233.5 + 50) = .096 \text{ mm}$$



#### 3

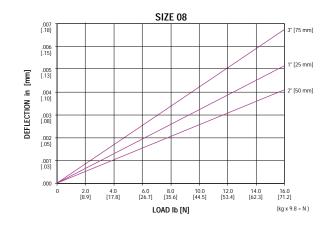
#### STATIC DEFLECTIONS IN YAW

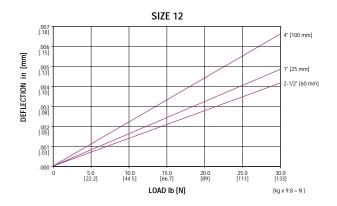
The graph below shows table yaw deflection due to static moment loads applied at distance Ah from bearing center with the unit extended.

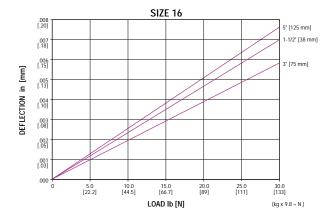


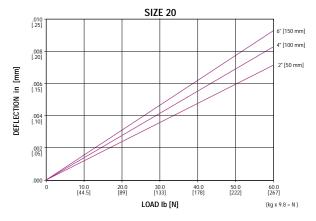
			MOMENT ARN		
	TRAVEL		Ah		
SIZE	in mm		in	mm	
	1	25	2.442	62.0	
08	2	50	3.830	97.3	
	3	75	4.914	124.8	
	1	25	2.717	69.0	
12	2-1/2	60	4.557	115.7	
	4	100	6.308	160.2	
	1-1/2	38	3.711	94.3	
16	3	75	5.049	128.2	
	5	125	7.292	185.2	
	2	50	4.286	108.9	
20	4	100	6.721	170.7	
	6	150	9.034	229.5	
	2	50	4.488	114.0	
25	4	100	6.811	173.0	
	6	150	9.194	233.5	

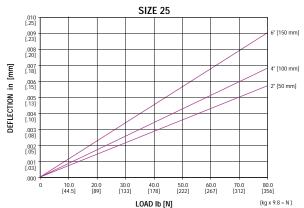
All tabulated and plotted values are typical and were determined empirically.











#### **IMPERIAL EXAMPLE:**

Determine the yaw deflection of a STPD125 x 6 slide at the center of gravity (CG) of a 10 lb load weight attached to the tool plate. The CG of the load is 2" further from the tool plate.

Calculate the moment of the application and the equivalent load at distance Ah.

Mp = Load x (Ah distance + CG distance)  
= 
$$10 \times (9.194 + 2) = 112 \text{ in-lb}$$

Equivalent load = (My / Ah) = 112 / 9.194 = 12 lb

Read the graph for a 12 lb load, deflection is approximately .0015".

Deflection at load = Deflection Ratio x (Ah + CG)  
= 
$$1.63 \times 10^{-4} \text{ x } (9.194 + 2) = .0018^{\circ}$$

#### **METRIC EXAMPLE:**

Determine the yaw deflection of a STPD525 x 150 slide at the center of gravity (CG) of a 45 N load weight attached to the tool plate. The CG of the load is 50 mm further from the tool plate.

Calculate the moment of the application and the equivalent load at distance Ah.

Equivalent load = 
$$(My / Ah) \times 1000 = 12.76 / 233.5 \times 1000 = 55 N$$

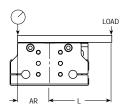
Read the graph for a 55 N load, deflection is approximately .04 mm.

Deflection at load = Deflection Ratio x (Ah + CG)  
= 
$$1.71 \times 10^{-4} \times (233.5 + 50) = .048 \text{ mm}$$



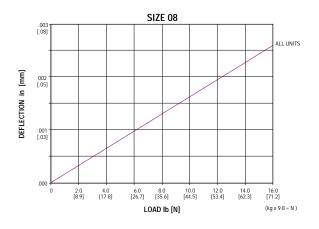
#### 3 STATIC DEFLECTION IN ROLL

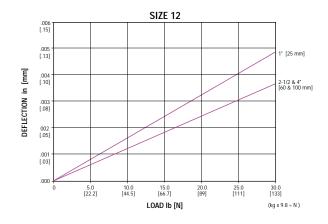
The graph on this page shows table roll deflection due to static moment loads applied at distance L from the center of the bearing. Values plotted in graphs were measured at point indicated.

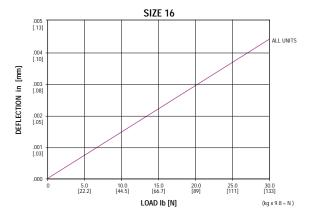


	TRAVEL			ANCE L	DISTANCE AR		
SIZE	in mm		in	mm	in	mm	
	1	25					
80	2	50	2	51	0.827	21.0	
	3	75					
	1	25					
12	2-1/2	60	2.5	64	1.042	26.5	
	4	100					
	1-1/2	38					
16	3	75	3.5	89	1.418	36.0	
	5	125					
	2	50					
20	4	100	4.5	114	1.515	38.5	
	6	150					
	2	50					
25	4	100	6	152	1.811	46.0	
	6	150					

All tabulated and plotted values are typical and were determined empirically.













## ENGINEERING DATA: SERIES STP SLIDES WITH RAIL BEARING

### **IMPERIAL EXAMPLE:**

Determine the roll deflection of a STPD125 x 6 slide at the center of gravity (CG) of a 10 lb load weight at 4" from the center of the slide.

Calculate the moment of the application and the equivalent load at distance L.

Mr = Load x Distance to CG of load = 10 x 4 = 40 in-lb

Equivalent load at L = Mr / L = 40 / 6 = 6.66 lb

Read the graph for a 6.7 lb load, deflection is approximately .0005". (This is at AR distance of 1.811)

Deflection Ratio = Deflection at AR / AR distance = .0005/1.811= 2.76 x 10<sup>-4</sup>

Deflection at load = Deflection Ratio x (CG distance) =  $2.76 \times 10^{-4} \times 4 = .0011$ "

#### METRIC EXAMPLE:

Determine the roll deflection of a STPD525 x 150 slide at the center of gravity (CG) of a 45 N load weight at 102 mm from center of the slide.

Calculate the moment of the application and the equivalent load at distance L.

Mr = Load x Distance to CG of load / 1000 = 45 x 102 / 1000 = 4.59 Nm

Equivalent load at L= (Mr / L) x 1000 = (4.59 / 152) x 1000 = 30.2 N

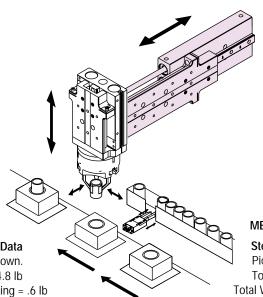
Read the graph for a 30.2 N load, deflection is approximately .013 mm. (This is at AR distance of 46 mm.)

Deflection Ratio = Deflection at AR / AR distance =  $.013 / 46 = 2.82 \times 10^{-4}$ 

Deflection at load = Deflection Ratio x (CG distance) =  $2.82 \times 10^{-4} \times 102 = .029 \text{ mm}$ 



## SLIDE SIZING EXAMPLE: SERIES STP SLIDES WITH RAIL BEARING



### **IMPERIAL**

### Step 1: Determine Application Data

Pick and place application as shown.

Total Weight of vertical slide = 4.8 lb

Total Weight of gripper and tooling = .6 lb

Total Weight of gripped object = .1 lb

Operating pressure = 80 psi

Required Travel = 5"

CG Dist = 1"

## Step 2: Determine the Total Weight of the system and the required thrust of the slide.

Calculate the Total Weight of the system:
Weight of attached slide = 4.8
Weight of gripper and tooling = .6
Weight of gripped object = .1
Total Weight = 5.5 lb

Since the application is horizontal, thrust calculation is not required at this step due to very low friction values.

Size 16 would be the minimum requirement based on the necessary travel.

## Step 3: Determine static and dynamic moment capacity First check size 16 for moment capacity.

From the Static Moment Chart for Yaw moment, Maximum yaw moment (My) for a 5" travel = 558 in-lb and Ah = 7.292"

 $My = (Ah + CG) \times LOAD (Total Weight)$ 

My Static =  $(7.292 + 1) \times 5.5 = 45.6$  in-lb, okay statically

My Dynamic = 558/10 = 55.8 in-lb, okay dynamically

Since Dynamic moment of the system is less than 55.8, the size 16 can be used.

### Step 4: Determine the amount of Deflection

From the yaw deflection graphs, determine the amount of deflection at the tool plate by using the Total Weight calculated above and finding the crossing point for a size  $16 \times 5$ .

Approximately .004 of deflection at the tool plate for this application.

**Note:** Dynamic forces from the attached slide and gripper can cause higher deflections than the value just calculated depending on deceleration methods.

Step 5: Calculate Stopping Capacity - see page 5A-35

#### METRIC

### Step 1: Determine Application Data

Pick and place application as shown.

Total Weight of vertical slide = 21.4 N

Total Weight of gripper and tooling = 2.7 N

Total Weight of gripped object = .4 N

Operating pressure = 5.5 bar Required Travel = 125 mm CG Dist = 25 mm

## Step 2: Determine the Total Weight of the system and the required thrust of the slide.

Calculate the Total Weight of the system:
Weight of attached slide = 21.4
Weight of gripper and tooling = 2.7
Weight of gripped object = .4

Total Weight = 24.5 N

Since the application is horizontal, thrust calculation is not required

at this step due to very low friction values.

Size 16 would be the minimum requirement based on the necessary travel.

## Step 3: Determine static and dynamic moment capacity First check size 16 for moment capacity.

From the Static Moment Chart for Yaw moment, Maximum yaw moment (My) for a 125 mm travel = 63 Nm and Ah = 185.2 mm

 $My = (Ah + CG) \times LOAD (Total Weight)$ 

My Static = (.1852 + .025) x 24.5 = 5.1 Nm, okay statically

My Dynamic = 63/10 = 6.3 Nm, okay dynamically

Since Dynamic moment of the system is less than 6.3, the size 16 can be used.

### Step 4: Determine the amount of Deflection

From the yaw deflection graphs, determine the amount of deflection at the tool plate by using the Total Weight calculated above and finding the crossing point for a size 16 x 125.

Approximately .10 mm of deflection at the tool plate for this application.

**Note:** Dynamic forces from the attached slide and gripper can cause higher deflections than the value just calculated depending on deceleration methods.

Step 5: Calculate Stopping Capacity - see page 5A-35





# TRAVEL ADJUSTMENT AND SHOCK PAD ON EXTEND IN POSITION 1

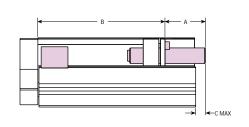
AE2

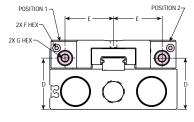
# TRAVEL ADJUSTMENT AND SHOCK PAD ON EXTEND IN POSITION 2

This option provides travel adjustment with a shock pad on extend in position 1. Shock pads provide excellent noise reduction and energy absorption capability. Travel on extend can be reduced by a maximum of 'A' shown in the table below. Adjust travel adjustment screw to the required position using 'G' hex wrench and lock into place using 'F' hex wrench. Refer to page 5A-35 for stopping capacity of the shock pad.

This option provides travel adjustment with a shock pad on extend in position 2. Shock pads provide excellent noise reduction and energy absorption capability. By using -AE1 and -AE2 options together, yaw moments are greatly reduced and may eliminate the need for a shock absorber. Travel on extend can be reduced by a maximum of 'A' shown in the table below. Adjust travel adjustment screw to the required position using 'G' hex wrench and lock into place using 'F' hex wrench. Refer to page 5A-35 for stopping capacity of the shock pad.

	TRA	/EL	A		В		(	2	D	)	E		F	G
SIZE	in	mm	in	mm	in	mm	in	mm	in	mm	in	mm	HEX	HEX
	1	25	.650	16.5	2.953	75.0	.591	15.0						
80	2	50	.827	21.0	3.779	96.0	_	_	.752	19.1	0.74	18.9	2 mm	3 mm
	3	75	.827	21.0	4.783	121.5	_							
	1	25	.749	19.0	2.755	70.0	.120	3.0						
12	2-1/2	60	.944	24.0	4.490	114.0	_	_	1.022	26.0	0.96	24.3	2.5 mm	3 mm
	4	100	1.122	28.5	6.081	154.5								
	1-1/2	38	.945	24.0	3.662	93.0	.039	1.0						
16	3	75	1.122	28.5	4.981	126.5	_	_	1.181	30.0	1.260	32	2.5 mm	5 mm
	5	125	1.102	28.0	6.989	177.5	_							
	2	50	1.281	32.5	4.152	105.5	_	_						
20	4	100	1.654	42.0	6.576	167.0	_	_	1.447	36.8	1.42	36	2.5 mm	6 mm
	6	150	1.299	33.0	8.896	226.0	_	_						
	2	50	1.437	36.5	4.487	114.0	.354	9.0						
25	4	100	1.181	30.0	6.732	171.0	-	_	1.810	46.0	1.71	43.5	3 mm	6 mm
	6	150	1.122	28.5	8.800	223.5	_	_						

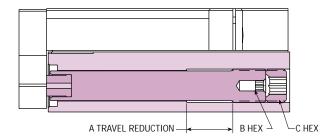




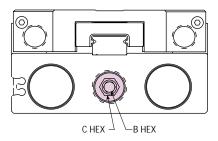


# TRAVEL ADJUSTMENT AND SHOCK PAD ON RETRACT

This option provides travel adjustment with a shock pad on retract. Shock pads provide excellent noise reduction and energy absorption capability. Travel on retract can be reduced by a maximum of 'A' shown in the table below. Adjust travel adjustment screw to the required position using 'B' hex wrench and lock into place using 'C' hex wrench. Refer to page 5A-35 for stopping capacity of the shock pad.



	Α		В	C
SIZE	in	mm	HEX	HEX
08	.512	13.0	2 mm	3 mm
12	.669	17.0	4 mm	5 mm
16	.984	25.0	5 mm	6 mm
20	1.063	27.0	6 mm	8 mm
25	1.063	27.0	6 mm	10 mm





### TRANSITIONAL FIT DOWEL **PIN HOLES**



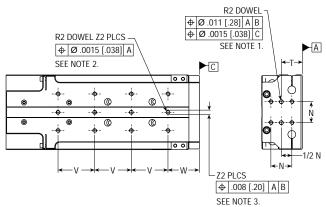
### PRECISION FIT DOWEL PIN HOLES

This option provides a compromise fit between clearance and interference. Transitional fits are used where accuracy of location is important, but a small amount of clearance is permissible.

This option provides a H7 tolerance precision fit with dowel pins. Precision fits are used where accuracy of location is of prime importance and for parts requiring rigidity and alignment.

LETTER			SIZ	E 08					SIZ	E 12				SIZ	ZE 16					SIZ	E 20					SIZ	E 25		
DIM	in	mm	in	mm	in	mm	in	mm	in	mm	in	mm	in mm	in	mm	in	mm	in	mm	in	mm	in	mm	in	mm	in	mm	in	mm
TRAVEL	1	25	2	50	3	75	1	25	2.5	60	4	100	1-1/2 38	3	75	5	125	2	50	4	100	6	150	2	50	4	100	6	150
L			1.102	28.0					1.496	38.0				1.929	49.0					2.205	56.0					2.677	68.0		
N			.551	14.0					.670	17.0				.905	23.0					1.024	26.0					1.260	32.0		
R1		3 n	nm x 2	.5 mm	DP			4 m	nm x 2	.5 mm	DP		5	mm x	3 mm [	)P			5	mm x	3 mm [	DP			6 n	nm x 5	.5 mm l	DP	
R2		3	mm x 3	3 mm D	)P			4 ו	mm x 4	mm D	Ρ		5	mm x	5 mm [	)P			5	mm x	5 mm [	DP			6	mm x	6 mm D	Ρ	
S2		3		3		4		3		3		4	3		3		4		3		3		4		3		3		4
T			.532	13.5					.713	18.1				.906	23.0					1.078	27.4					1.260	32.0		
V			.868	22.0					1.102	28.0				1.496	38.0					1.654	42.0					2.205	56.0		
W			1.102	28.0					1.300	33.0				1.378	35.0					1.851	47.0					1.890	48.0		
Z2		2		3		4		2		3		4	2		3		4		2		3		4		2		3		4

Ø R DOWEL		TOLERANCE	
HOLE	STANDARD	J3 OPTION	J8 OPTION
3 mm	+.0004/0009	+.0013/+.0003	+.0004/0000
3 11111	[+.010/024]	[+.033/+.008]	[+.010/000]
4 mm	+.0004/0009	+.0015/+.0005]	+.0005/0000
4 111111	[+.010/024]	[+.033/+.008]	[+.010/000]
5 mm	+.0004/0009	+.0015/+.0004]	+.0005/0000
3 111111	[+.010/024]	[+.038/+.010]	[+.012/000]
6 mm	+.0004/0009	+.0015/+.0005]	+.0005/0000
0 111111	[+.010/024]	[+.038/+.013]	[+.012/000]

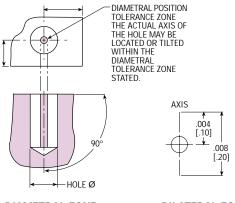


## R1 DOWEL S2 PLCS Ф Ø .0015 [.038] A B SEE NOTE 4 0 0 -N-

NOTES:  $\Phi$  = POSITION TOLERANCE, THIS TOLERANCE DETERMINES THE LOCATION OF THE HOLES AND THE PERPENDICULARITY TO THE INDICATED DATUM FEATURES.

- 1) THE AXIS OF THESE DOWEL HOLES ARE LOCATED TO SURFACE A (DATUM) AND DOWEL HOLE PATTERN B (DATUM) WITHIN A .011 DIAMETRAL TOLERANCE ZONE. ADDITIONALLY THE AXIS OF THE HOLES ARE LOCATED TO EACH OTHER AND PERPENDICULAR TO SURFACE C (DATUM) WITHIN A.0015 DIAMETRAL TOLERANCE ZONE.

- 10 ILERANICE ZONE.
  2) THE AXIS OF THE DOWEL HOLES ARE LOCATED TO EACH OTHER AND PERPENDICULAR TO SURFACE
  A (DATUM) WITHIN A .0015 [.038] DIAMETRAL TOLERANCE ZONE.
  3) THE AXIS OF THESE HOLES ARE LOCATED TO SURFACE A (DATUM) AND DOWEL HOLE PATTERN B
  (DATUM) WITHIN .008 BILATERAL TOLERANCE ZONE.
  4) THE AXIS OF THESE DOWEL HOLES ARE LOCATED TO EACH OTHER AND PERPENDICULAR TO SURFACE
  A (DATUM) WITHIN A .0015 [.038] DIAMETRAL TOLERANCE ZONE.



**DIAMETRAL ZONE** 

**BILATERAL ZONE** 

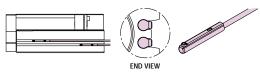




### MAGNET FOR PHD SERIES 6790 MINIATURE REED AND SOLID STATE SWITCHES

This option equips the unit with a magnetic piston for use with PHD's Series 6790 Switch. The switch housing is completely contained by the slide housing and provides a very compact switch design. The switches mount easily into two small grooves located on the side of the slide housing and are locked into place with a set screw.

ı	DADT NO	DECCRIPTION
	PART NO.	DESCRIPTION
	67902-1-02	NPN (Sink) or PNP (Source) DC Reed, 2 m cable
	67902-1-05	NPN (Sink) or PNP (Source) DC Reed, 5 m cable
	67903-1-02	NPN (Sink) DC Solid State, 2 m cable
	67903-1-05	NPN (Sink) DC Solid State, 5 m cable
	67904-1-02	PNP (Source) DC Solid State, 2 m cable
	67904-1-05	PNP (Source) DC Solid State, 5 m cable
	67922-1	NPN (Sink) or PNP (Source) DC Reed, Quick Connect
	67923-1	NPN (Sink) DC Solid State, Quick Connect
	67924-1	PNP (Source) DC Solid State, Quick Connect
	63549-02	2 m Cordset with Quick Connect
ı	63549-05	5 m Cordset with Quick Connect
-		



### **REED BENEFITS**

- Available as 4.5 30 VDC model for simple interfacing to sequencers and programmable controllers.
- Can be used to directly drive some types of relays or valve solenoids within the switch specifications stated.

SPECIFICATIONS	67902 & 67922
OPERATING PRINCIPLE	Magnetic Reed
ACTUATED BY	Piston Magnet
INPUT VOLTAGE	4.5 - 30 VDC
OUTPUT TYPE	Contact Closure
CURRENT RATING	50 mA Max.
CONTACT RESISTANCE	.115 Ohm Max.
ENVIRONMENTAL	IP67
OPERATING TEMP.	-20° to 85°C

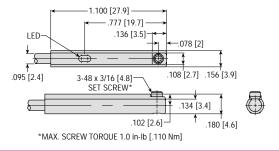
		HYSTERESIS	BAND WIDTH
SIZE	REPEATABILITY	MAXIMUM	MINIMUM/MAXIMUM
80	± .005 [± .13]	.060 [1.5]	.360 / .690 [9.1 / 17.5]
12	± .005 [± .13]	.080 [2.0]	.230 / .340 [5.8 / 8.6]
16	± .005 [± .13]	.075 [1.9]	.340 / .440 [8.6 / 11.2]
20	± .005 [± .13]	.085 [2.2]	.130 / .360 [3.3 / 9.1]
25	± .004 [± .10]	.070 [1.8]	.300 / .425 [7.6 / 10.8]

### **SOLID STATE BENEFITS**

- Solid state switches afford long life. Constant amplitude output allows use with most digital logic systems.
- Switch circuitry protects against voltage surges and other electrical anomalies associated with operating systems.
- Excellent switch hysteresis characteristics and symmetry.
- Offered in 4.5 30 VDC current sinking and current sourcing versions for simple interfacing to electronic system controllers.

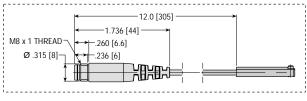
SPECIFICATIONS	67903 & 67923	67904 & 67924
OPERATING PRINCIPLE	Solid	l State
ACTUATED BY	Piston	Magnet
INPUT VOLTAGE	4.5 - 3	30 VDC
OUTPUT TYPE	NPN (Sink)	PNP (Source)
CURRENT RATING	50 m.	A. Max
VOLTAGE DROP	.5 '	VDC
SWITCH BURDEN	10 m/	A. Max.
ENVIRONMENTAL	l F	P67
OPERATING TEMP.	-20° t	o 85°C

		HYSTERESIS	BAND '	WIDTH
SIZE	REPEATABILITY	MAXIMUM	MINIMUM/	MAXIMUM
80	± .007 [± .18]	.065 [1.7]	.320 / .580	[8.1 / 14.7]
12	± .007 [± .18]	.095 [2.4]	.300 / .450	[7.8 / 11.5]
16	± .007 [± .18]	.095 [2.4]	.330 / .510	[8.4 / 13.0]
20	± .005 [± .13]	.110 [2.8]	.190 / .380	[4.8 / 9.6]
25	± .005 [± .13]	.080 [2.0]	.320 / .470	[8.1 / 11.9]

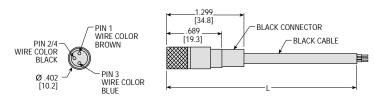


### MALE QUICK CONNECT DETAIL

METRIC INFORMATION SHOWN IN [ ]



#### 63549-xx CORDSET WITH FEMALE QUICK CONNECT



	LETTER DIM.
MODEL NO.	L
63549-02	78.74 [2 m]
63549-05	196.85 [5 m]



### SOLID STATE WIRING SCHEMATICS

MODEL NO. 67903-1 & 67923-1 - NPN (SINK)

**INPUT:** 4.5 - 30 VDC

LOAD CURRENT: 50 mA. MAX. SWITCH HOUSING COLOR: BLACK (Bi-polar LED emits a yellow light)

### **CABLED MODEL 67903**



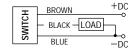
#### **QUICK CONNECT MODEL 67923**





MODEL NO. 67904-1 & 67924-1 - PNP (SOURCE) INPUT: 4.5-30 VDC LOAD CURRENT: 50 mA. MAX. SWITCH HOUSING COLOR: BLACK (Bi-polar LED emits a red light.)

#### **CABLED MODEL 67904**



#### **QUICK CONNECT MODEL 67924**





### REED WIRING SCHEMATICS

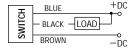
MODEL NO. 67902-1 & 67922-1 - NPN (SINK) OR PNP (SOURCE)

**INPUT:** 4.5 - 30 VDC

LOAD CURRENT: 50 mA. MAX. SWITCH HOUSING COLOR: BLACK

(LED emits a red light)

#### CABLED MODEL 67902 - NPN (SINK)

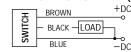


### QUICK CONNECT MODEL 67922 - NPN (SINK)





#### CABLED MODEL 67902 - PNP (SOURCE)



### QUICK CONNECT MODEL 67922 - PNP (SOURCE)







# SHOCK ABSORBER INSTALLED ON EXTEND IN POSITION 1

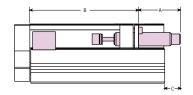
This option provides shock absorbers and travel adjustment on extend in position 1. Travel on extend can be reduced by a maximum of 'A' shown in the table below. Adjust shock absorber screw to the required position using a large screwdriver and lock into place using 'F' hex wrench. Refer to page 5A-36 for shock absorber selection requirements.

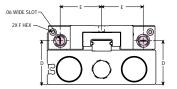
	TRA	/EL	A		В		(	2	D	)	E		F
SIZE	in	mm	in	mm	in	mm	in	mm	in	mm	in	mm	HEX
	1	25	.650	16.5	2.953	75.0	.591	15.0					
08	2	50	.827	21.0	3.779	96.0	_	-	.752	19.1	0.743	18.9	2 mm
	3	75	.827	21.0	4.783	121.5							
	1	25	1.064	27.0	2.755	70.0	.433	11.0					
12	2-1/2	60	0.828	21.0	4.490	114.0	_	_	1.022	26.0	0.96	24.4	2.5 mm
	4	100	0.866	22.5	6.081	154.5	-	_					
	1-1/2	38	.945	24.0	3.662	93.0	.039	1.0					
16	3	75	1.122	28.5	4.981	126.5	-	_	1.181	30.0	1.260	32	2.5 mm
	5	125	1.102	28.0	6.989	177.5	-	_					
	2	50	1.280	32.5	4.152	105.5	-	_					
20	4	100	1.280	32.5	6.576	167.0	_	-	1.447	36.8	1.42	36	2.5 mm
	6	150	1.280	32.5	8.896	226.0	_	_					
	2	50	1.772	45.0	4.487	114.0	.669	17.0					
25	4	100	1.516	38.5	6.732	171.0	_	_	1.810	46.0	1.712	43.5	3 mm
	6	150	1.457	37.0	8.800	223.5	_	-					



# SHOCK ABSORBER INSTALLED ON EXTEND IN POSITION 2

This option provides shock absorbers and travel adjustment on extend in position 2. Travel on extend can be reduced by a maximum of 'A' shown in the table below. Adjust shock absorber screw to the required position using a large screwdriver and lock into place using 'F' hex wrench. Refer to page 5A-36 for shock absorber selection requirements.







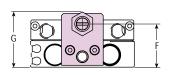
### FOR SIZE 08 ONLY

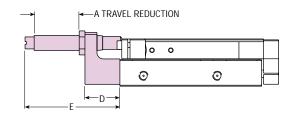


# SHOCK ABSORBER INSTALLED ON RETRACT

This option provides shock absorbers and travel adjustment on retract. Travel on retract can be reduced by a maximum of 'A' shown in the table at right. Adjust travel to the required position using a large screwdriver and lock into place using 11 mm hex wrench. Refer to page 5A-36 for shock absorber selection requirements.

F	1		D				F	G		
in	mm	in	mm	in	mm	in	mm	in	mm	
.905	23.0	.728	18.5	2.008	51.0	.901	22.9	1.151	29.2	





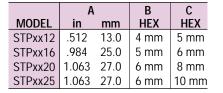
### FOR SIZES 12, 16, 20, & 25

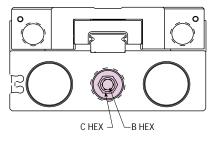


# SHOCK ABSORBER INSTALLED ON RETRACT

This option provides shock absorbers and travel adjustment on retract. Travel on retract can be reduced by a maximum of 'A' shown in the table below. Adjust travel to the required position using 'B' hex wrench and lock into place using 'C' hex wrench. Refer to page 5A-36 for shock absorber selection requirements.

<u> </u>	A TRAVEL REDUCTION B HEX 2 C HE	EX







# CORROSION RESISTANT GUIDE SHAFTS

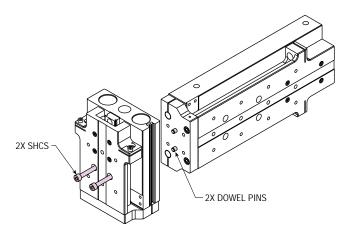
This option provides stainless steel guide shafts with hard chrome plating, for use in applications where the standard shaft ends may corrode.



## **ACCESSORIES:** SERIES STP SLIDES WITH RAIL BEARING

### MODULAR MOUNTING KITS

Modular design of the Series STP housings and tool plates allow slide units to bolt and dowel together without the need for a transition plate. See chart for slide compatibility and hardware kits required. Each kit contains 2 dowel pins and 2 SHCS to mount the units together. Series STP units can also be bolted directly together as shown. PHD recommends that a -J3 option (transitional fit) be specified with the slide ordering data to allow the units to dowel together properly.



		KIT NUMBERS				
PRIMARY	SECONDARY	IMPERIAL	METRIC			
STPDx08	STPDx08	68125-01	68125-02			
STPDx12	STPDx12	70770-01	70770-02			
STPDx16	STPDx16	68053-01	68053-02			
STPDx20	STPDx20	70870-01	70870-02			
STPDx25	STPDx25	68043-01	68043-02			

## STOPPING CAPACITY: SERIES STP SLIDES WITH RAIL BEARING

### STOPPING CAPACITY SELECTION

To determine stopping capacity, calculate total moving weight. From Table 1, determine slide standard moving weight, add any additional weight adders due to options and add attached payload. This will be total moving weight  $W_{\text{TM}}$ .

Example: STPD125 x 2 -AE1-AE2 with 10 lb load [STPD525 x 50-AE1-AE2 with 44.5 N load]

WTM = 2.6 lb + .29 lb + .29 lb + 10 lb = 13.18 lb [11.6 N + 1.29 N + 1.29 N + 44.5 N = 58.68 N]

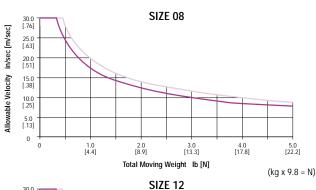
Using the Kinetic Energy Graphs below, plot the total moving weight against impact velocity. If the value plotted is below the curve, then shock pads are an adequate deceleration method. If it is above the curve, hydraulic shock absorbers are required.

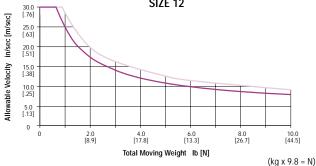
To determine the correct hydraulic shock absorber, complete the calculations on the next page.

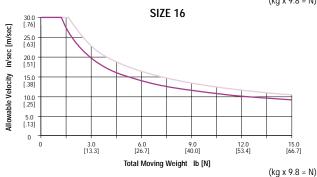
TABLE 1

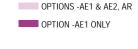
	TRA	VEL	STP MOVING WEIGHT		WEIGHT ADDERS -AE1, -AE2, -NE1x, -NE2x		PISTON AREA EXTEND		PISTON AREA RETRACT	
SIZE	in	mm	lb	N	lb	N	in²	mm <sup>2</sup>	in <sup>2</sup>	mm <sup>2</sup>
	1	25	0.24	1.1						
80	2	50	0.36	1.6	0.06	0.27	0.16	101	0.12	75
	3	75	0.40	1.8						
	1	25	0.42	1.9						
12	2-1/2	60	0.60	2.7	0.09	0.42	0.35	226	0.26	170
	4	100	0.78	3.4						
	1-1/2	38	0.9	4.0						
16	3	75	1.1	4.9	0.13	0.58	0.62	402	0.47	302
	5	125	1.4	6.2						
	2	50	1.4	6.2						
20	4	100	1.9	8.5	0.20	0.91	0.97	628	0.73	471
	6	150	2.4	10.7						
	2	50	2.6	11.6						
25	4	100	3.6	16.0	0.29	1.29	1.52	982	1.17	756
	6	150	4.3	19.1						

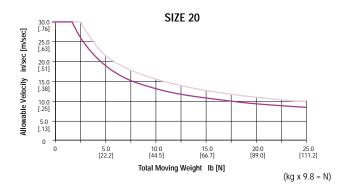
### MAXIMUM ALLOWABLE KINETIC ENERGY GRAPHS FOR SHOCK PADS

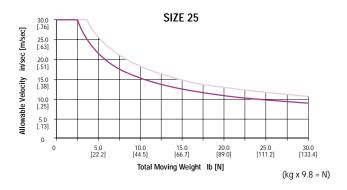












## SHOCK ABSORBER SELECTION GUIDE: SERIES STP

#### PHD SHOCK ABSORBER SPECIFICATIONS CHART

	PHD SHOCK ABSORBER	STROKE		THREAD			E <sub>T</sub> C TOTAL ENERGY PER HOUR		MAX PRO	PELLING RCE
SIZE	NO.	in	m	TYPE	in-lb	Nm	in-lb	Nm	lb	N
08 &12	68149-01-x	.210	.0053	M8 x1	20	2.26	50,000	5654	45	200
16	68015-01-x	.240	.0061	M10 x 1	40	4.52	110,000	12439	80	356
20	70861-01-x	.400	.0102	M12 x 1	65	7.35	250,000	28269	120	534
25	67127-01-x	.448	.0114	M14 x 1.5	135	15.26	260,000	29400	200	890

### SHOCK ABSORBER SIZING CALCULATION:

Follow the next six steps to size shock absorbers.

STEP 1: Identify the following parameters. These must be known for all energy absorption calculations. Variations or additional information may be required in some cases.

- A. The total moving weight (WTM) to be stopped. (completed from prior page)
- B. The slide velocity (V) at impact with the shock absorber.
- C. Number of cycles per hour.
- D. Orientation of the application's motion (i.e. horizontal or vertical application). See the next two pages.
- E. Operating pressure

**STEP 2:** Calculate the kinetic energy of the total moving weight.

EK (in-lb) = 
$$.5 \times \frac{W_{TM}}{386} \times V^2$$
 EK (Nm) =  $.5 \times \frac{W_{TM}}{9.8} \times V^2$ 

Note: WTM in kg mass may

 $E\kappa$  (Nm) = .5 x W<sub>TM</sub> x V<sup>2</sup>

be substituted for WTM

STEP 3: Calculate the propelling force (FD) for both extend and retract. Refer to previous page for Effective Piston Areas.

Horizontal application: FD = Effective Piston Area x P Vertical application: FD = (Effective Piston Area x P) ± WTM + indicates working with gravity, - indicates working against gravity Note: when using mm<sup>2</sup> and bar units, it will be necessary to multiply the Effective Piston Area x P by a factor of .1 to obtain the correct unit of measure.

Use Shock Absorber Specification Chart to verify that the selected unit has an FG capacity greater than the value just calculated. If not, select a larger shock absorber or slide.

Calculate the work energy input (Ew = FD x S) using the travel of the shock absorber selected.

**STEP 4:** Calculate the total energy.  $E_T = E_K + E_W$ 

Use Shock Absorber Specification Chart to verify that the selected unit has an E<sub>T</sub> capacity greater than the value just calculated. If not, select a larger shock absorber or slide.

STEP 5: Calculate the total energy that must be absorbed per hour (ETC). ETC = ET  $\times$  C

Use Shock Absorber Specification Chart to verify that the selected unit has an E<sub>T</sub>C capacity greater than the value just calculated. If not, select a larger shock absorber or slide.

STEP 6: Determine the damping constant for the selected shock absorber. Using the appropriate Shock Absorber Performance Graph, locate the intersection point for impact velocity (V) and total energy (E<sub>T</sub>). The area (-2 or -3) that the point falls in is the correct damping constant for the application.

### SYMBOLS DEFINITIONS

= Number of cycles per hour

= Cylinder bore diameter inch [mm]

Kinetic energy in-lb [Nm]

Total energy per cycle, E<sub>K</sub> + E<sub>W</sub> in-lb [Nm]

= Total energy per hour in-lb/hr [Nm/hr]

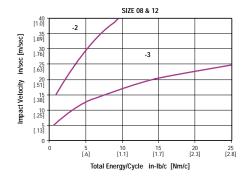
= Work or drive energy in-lb [Nm] = Propelling force lb [N]

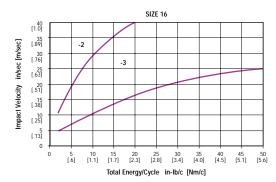
= Max Propelling force lb [N]

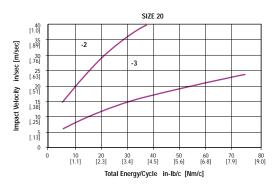
Operating pressure psi [bar]

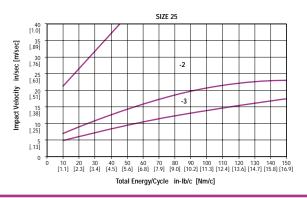
= Stroke of shock absorber inch [m] = Impact velocity in/sec [m/sec]

= Total moving weight lb [N or kg]





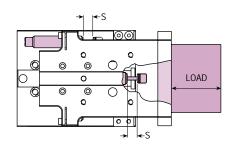






## SHOCK ABSORBER SELECTION GUIDE: SERIES STP

### SIZING EXAMPLE: HORIZONTAL APPLICATION



### **IMPERIAL**

#### STEP 1: Application Data

Example: STPD125 x 6 -NEx-NRx with a 20 lb payload on extend and 1 lb on retract.

A) W<sub>TM</sub> = Total moving weight = std moving + option adder + load Extend = 2.6 lb + .29 lb + 20 lb = 22.89 lb Retract = 2.6 lb + .29 lb + 1 lb = 3.89 lb

B) Velocity at impact: VE = 15 in/sec (extend), VR = 20 in/sec (retract)

C) Number of cycles/hour: C = 800 cycles/hr

D) Application type: HorizontalE) Operating pressure: 80 psi

#### STEP 2: Calculate the kinetic energy

 $E_K = .5 \times W_{TM} \times V^2 / 386$ 

Extend =  $.5 \times 22.89 \times 15^2 / 386 = 6.67$  in-lb Retract =  $.5 \times 3.89 \times 20^2 / 386 = 2.02$  in-lb

### STEP 3: Calculate the propelling force and work energy

FD= Effective Piston Area x Operating Pressure

Extend =  $1.52 \times 80 = 121.6 \text{ lb}$ Retract =  $1.17 \times 80 = 93.6 \text{ lb}$ 

Use the Shock Absorber Specification Chart to verify that the selected unit has an F<sub>G</sub> capacity greater than the value just calculated.

 $Ew = F_D \times S$ 

Extend = 121.6 x .448 = 54.5 in-lb Retract = 93.6 x .448 = 41.9 in-lb

### STEP 4: Calculate the total energy: ET = EK + EW

Extend = 6.67 + 54.5 = 61.17 in-lb Retract = 2.02 + 41.9 = 43.92 in-lb

Use the Shock Absorber Specification Chart to verify that the selected unit has an  $E_T$  capacity greater than the value just calculated.

### STEP 5: Calculate the total energy per hour: ETC = ET x C

Extend = 61.17 x 800 = 48,397 in-lb/hr Retract = 43.92 x 800 = 35,136 in-lb/hr

Use the Shock Absorber Specification Chart to verify that the selected unit has and E<sub>T</sub>C capacity greater that the value calculated.

### STEP 6: Determine the damping constant required

Using the appropriate Shock Absorber performance graph, locate the intersection point for impact velocity (V) and total energy ( $E_T$ ). The area (-2 or -3) that the point falls in is the correct damping constant for the application.

Unit should be ordered with -NE3-NR2 options or select shock 67127-01-3 for extend and shock 67127-01-2 for retract.

#### METRIC

#### STEP 1: Application Data

Example: STPD525 x 150 -NEx-NRx with a 89 N payload on extend and 4.4 N on retract.

A)  $W_{TM}$  = Total moving weight = std moving + option adder + load Extend = 11.6 N + 1.29 N + 89 N = 101.89 N Retract = 11.6 N + 1.29 N + 4.4 N = 17.29 N

B) Velocity at impact:  $V_E = .381$  m/sec (extend),  $V_R = .51$  m/sec (retract)

C) Number of cycles/hour: C = 800 cycles/hr

D) Application type: HorizontalE) Operating pressure: 5.5 bar

### STEP 2: Calculate the kinetic energy

 $E_K = .5 \times W_{TM} \times V^2 / 9.8$ 

Extend =  $.5 \times 101.89 \times .381^2 / 9.8 = .75 \text{ Nm}$ Retract =  $.5 \times 17.29 \times .51^2 / 9.8 = .23 \text{ Nm}$ 

### STEP 3: Calculate the propelling force and work energy

FD = Effective Piston Area x Operating Pressure x .1

Extend = 982 x 5.5 x .1 = 540 N Retract = 756 x 5.5 x .1 = 416 N

Use the Shock Absorber Specification Chart to verify that the selected unit has an FG capacity greater than the value just calculated.

 $Ew = Fd \times S$ 

Extend = 540 x .0114 = 6.16 Nm Retract = 416 x .0114 = 4.74 Nm

### STEP 4: Calculate the total energy: ET = EK + EW

Extend = .75 + 6.16 = 6.91 Nm Retract = .23 + 4.74 = 4.97 Nm

Use the Shock Absorber Specification Chart to verify that the selected unit has an  $E_T$  capacity greater than the value just calculated.

### STEP 5: Calculate the total energy per hour: ETC = ET x C

Extend =  $6.91 \times 800 = 5,531 \text{ Nm/hr}$ Retract =  $4.97 \times 800 = 3,976 \text{ Nm/hr}$ 

Use the Shock Absorber Specification Chart to verify that the selected unit has and  $E_TC$  capacity greater that the value calculated.

### STEP 6: Determine the damping constant required

Using the appropriate Shock Absorber performance graph, locate the intersection point for impact velocity (V) and total energy ( $E_T$ ). The area (-2 or -3) that the point falls in is the correct damping constant for the application.

Unit should be ordered with -NE3-NR2 options or select shock 67127-01-3 for extend and shock 67127-01-2 for retract.



## SHOCK ABSORBER SELECTION GUIDE: SERIES STP

### SIZING EXAMPLE: VERTICAL APPLICATION

### **IMPERIAL**

### STEP 1: Application Data

Example: STPD125 x 2 -AE1-NE1x-NRx with a 30 lb payload on extend and 1 lb on retract

A) W<sub>TM</sub> = Total moving weight = std moving + option adder + load

Extend = 2.6 lb + .29 lb + .29 lb + 30 lb = 33.18 lb Retract = 2.6 lb + .29 lb + .29 lb + 1 lb = 4.18 lb

B) Velocity at impact: VE = 25 in/sec (extend), VR = 20 in/sec (retract)

C) Number of cycles/hour: C = 800 cycles/hr

D) Application type: VerticalE) Operating pressure: 80 psi

### STEP 2: Calculate the kinetic energy

 $E_K = .5 x W_{TM} x V^2 / 386$ 

Extend =  $.5 \times 33.18 \times 25^2 / 386 = 26.9 \text{ in-lb}$ 

Retract =  $.5 \times -4.18 \times 20^2 / 386 = -2.2$  in-lb (working against gravity)

Note: -AR option could replace -NRx option

### STEP 3: Calculate the propelling force and work energy

FD= (Effective Piston Area x Operating Pressure)  $\pm$  WTM Extend = (1.52 x 80) + 30 = 151.6 lb (working with gravity) Retract = (1.17 x 80) - 4.18 = 89.42 lb (working against gravity)

Use the Shock Absorber Specification Chart to verify that the selected unit has an F<sub>G</sub> capacity greater than the value just calculated.

 $Ew = F_D x S$ 

Extend = 151.6 x .448 = 67.9 in-lb Retract = 89.42 x .448 = 40.1 in-lb

### STEP 4: Calculate the total energy: $E_T = E_K + E_W$

Extend = 26.9 + 67.9 = 94.8 in-lb Retract = -2.2 + 40.1 = 37.9 in-lb

Use the Shock Absorber Specification Chart to verify that the selected unit has an  $E_{\text{T}}$  capacity greater than the value just calculated.

### STEP 5: Calculate the total energy per hour: ETC = ET x C

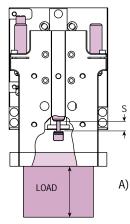
Extend = 94.8 x 800 = 75,840 in-lb/hr Retract = 37.9 x 800 = 30,320 in-lb/hr

Use the Shock Absorber Specification Chart to verify that the selected unit has and E<sub>T</sub>C capacity greater that the value calculated.

### STEP 6: Determine the damping constant required

Using the appropriate Shock Absorber performance graph, locate the intersection point for impact velocity (V) and total energy ( $E_T$ ). The area (-2 or -3) that the point falls in is the correct damping constant for the application.

Unit should be ordered with -NE12-NR2 options or select shock 67127-01-2 for extend and shock 67127-01-2 for retract.



### **METRIC**

### STEP 1: Application Data

Example: STPD525 x 50 -AE1-NE1x-NRx with a 133 N payload on extend and 4.4 N on retract

Extend = 11.6 N + 1.29 N + 1.29 N + 133 N = 147.18 N Retract = 11.6 N + 1.29 N + 1.29 N + 4.4 N = 18.58 N

B) Velocity at impact: V<sub>E</sub> = .64 m/sec (extend), V<sub>R</sub> = .51 m/sec (retract)

C) Number of cycles/hour: C = 800 cycles/hr

D) Application type: VerticalE) Operating pressure: 5.5 bar

### STEP 2: Calculate the kinetic energy

 $EK = .5 \times W_{TM} \times V^2 / 9.8$ 

Extend =  $.5 \times 147.18 \times .64^2 / 9.8 = 3.08 \text{ Nm}$ 

Retract =  $.5 \times -18.58 \times .51^2 / 9.8 = -.25 \text{ Nm}$  (working against gravity)

Note: -AR option could replace -NRx option

### STEP 3: Calculate the propelling force and work energy

F<sub>D</sub>= (Effective Piston Area x Operating Pressure x .1)  $\pm$  W<sub>TM</sub> Extend = (982 x 5.5 x .1) + 147.18 N = 673 N (working with gravity) Retract = (756 x 5.5 x .1) - 18.58 N = 397 N (working against gravity)

Use the Shock Absorber Specification Chart to verify that the selected unit has an  ${\sf FG}$  capacity greater than the value just calculated.

 $Ew = F_D x S$ 

Extend =  $673 \times .0114 = 7.67 \text{ Nm}$ Retract =  $397 \times .0114 = 4.53 \text{ Nm}$ 

### STEP 4: Calculate the total energy: ET = EK + EW

Extend = 3.08 + 7.67 = 10.75 Nm Retract = -.25 + 4.53 = 4.28 Nm

Use the Shock Absorber Specification Chart to verify that the selected unit has an  $E_{\text{T}}$  capacity greater than the value just calculated.

### STEP 5: Calculate the total energy per hour: ETC = ET x C

Extend = 10.75 x 800 = 8600 Nm/hr Retract = 4.28 x 800 = 3424 Nm/hr

Use the Shock Absorber Specification Chart to verify that the selected unit has and E<sub>T</sub>C capacity greater that the value calculated.

#### STEP 6: Determine the damping constant required

Using the appropriate Shock Absorber performance graph, locate the intersection point for impact velocity (V) and total energy ( $E_T$ ). The area (-2 or -3) that the point falls in is the correct damping constant for the application.

Unit should be ordered with -NE12-NR2 options or select shock 67127-01-2 for extend and shock 67127-01-2 for retract.

